

## NO VENT TANK FILL AND TRANSFER LINE CHILLDOWN ANALYSIS BY GENERALIZED FLUID SYSTEM SIMULATION PROGRAM (GFSSP)

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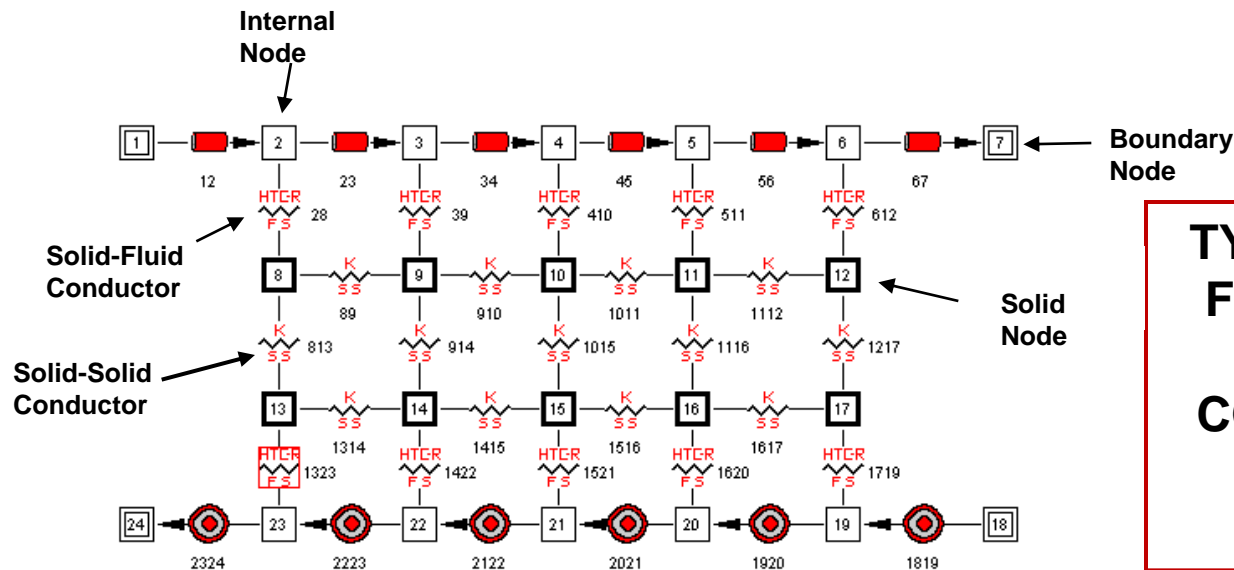
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TFAWS 2013  
July 29-August 2, 2013  
Kennedy Space Center  
KSC, FL



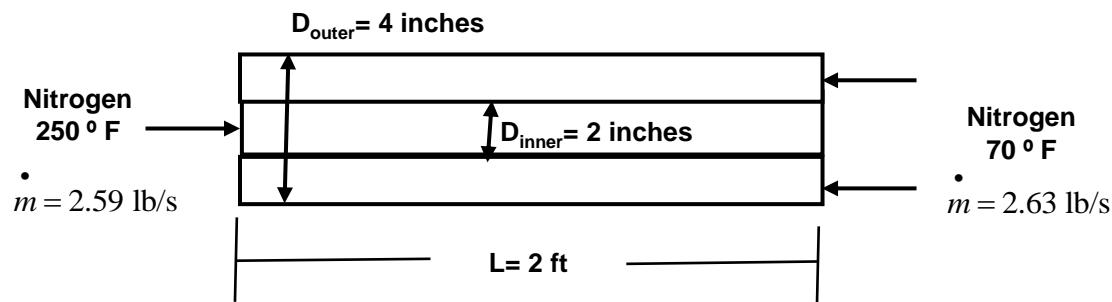
# Introduction



*GFSSP is a finite volume based Thermo-fluid system analysis program developed at NASA*



**TYPICAL GFSSP  
FLOW CIRCUIT  
FOR A  
COUNTERFLOW  
HEAT  
EXCHANGER**

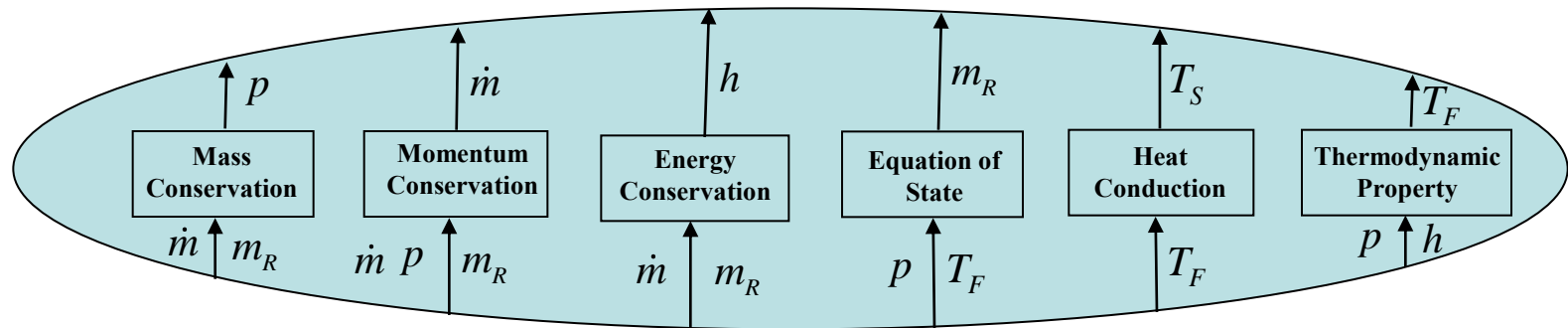




# MATHEMATICAL FORMULATION



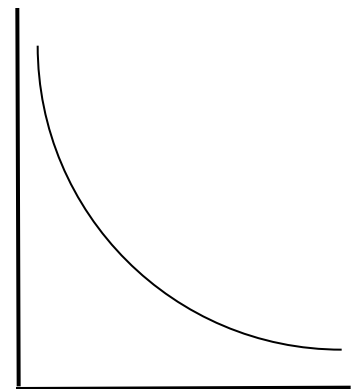
- GFSSP uses a pressure based method
- GFSSP's solver uses a combination of simultaneous and successive substitution method



**GFSSP Iteration Cycle**

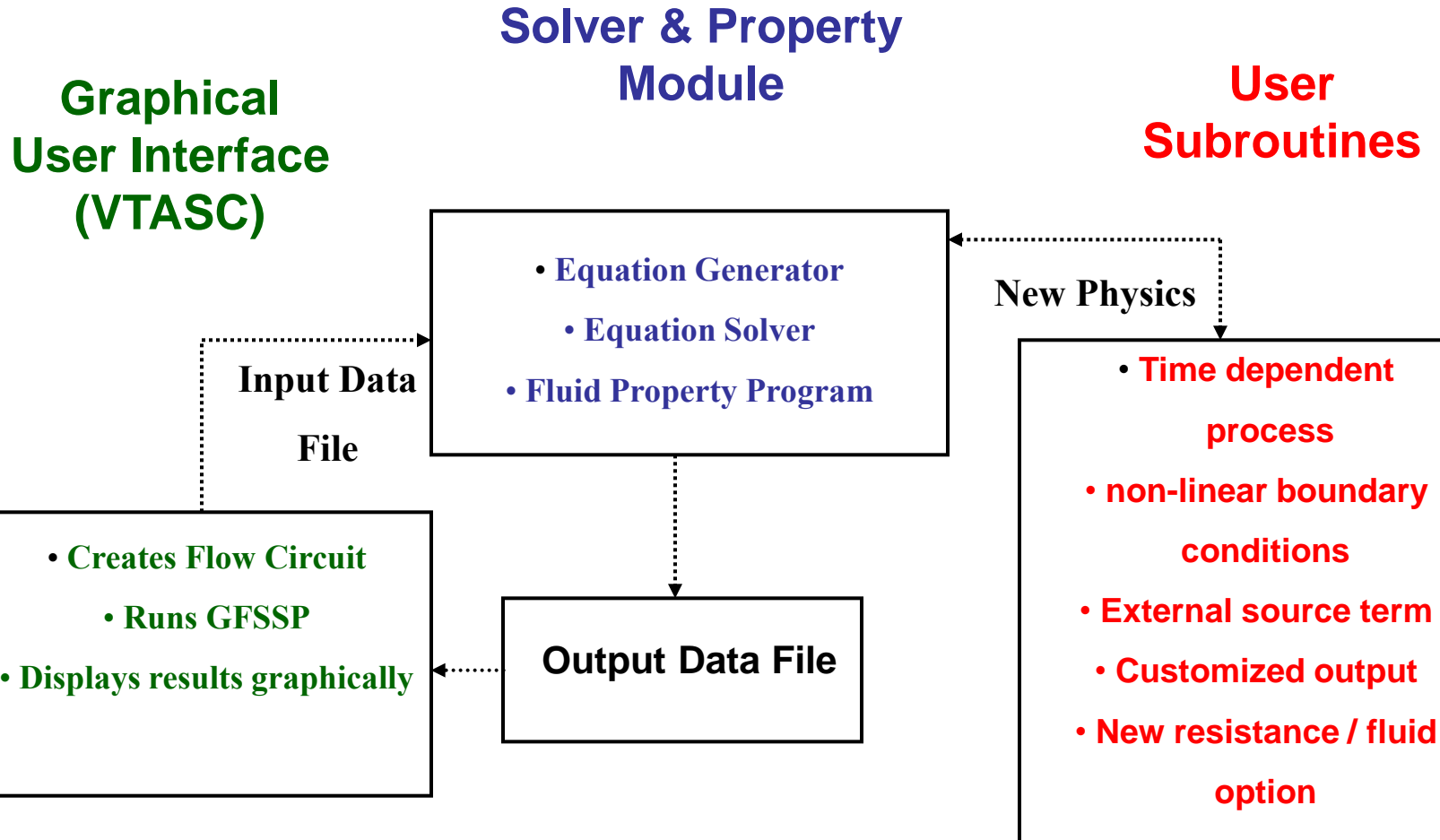
$p$	- Pressure	$\dot{m}$	- Flowrate	$T_F$	- Fluid Temperature
$m_R$	- Resident Mass	$h$	- Enthalpy	$T_S$	- Solid Temperature

**Error**





# PROGRAM STRUCTURE



• GFSSP has integrated two Thermo-dynamic Property Programs, GASP/WASP & GASPAK



# Capabilities



- Steady and unsteady flow
- Compressible and incompressible flow
- Single fluid and mixture
- 25 flow resistance 33 fluid options
- Options for new components through User Subroutine
- Options Pressure Regulator, Flow Regulator, Control Valve
- Fluid Transients (e.g. Water hammer)
- Conjugate Heat Transfer
- Options for Model Enhancement through User Subroutine
- Multi-Dimensional Flow Model in Fluid Network System



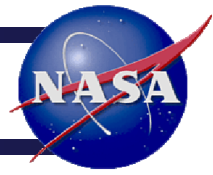
# No Vent Chill & Fill Model of Cryogenic Tank



- Tank Chillydown in micro-gravity environment is different than ground based tank chillydown
- During normal gravity chillydown, a vent on top of the tank is kept open to vent the vapor generated during chill process and maintain a low tank pressure
- In micro-gravity environment, due to absence of stratification, such practice may result in dumping large amount of liquid overboard
- The intent of no-vent chill & fill method is to minimize the loss of propellant during chillydown of propellant tank in micro-gravity environment
- No-vent chill & fill method consists of repeated cyclic process of charge, hold and vent
- The purpose of this presentation is to report the progress of an analytical effort to develop a simulation model of no-vent chill & Fill method for loading a cryogenic tank in space



# K-site Test Tank(GRC) - The LH2 chilldown test



**Tank Material – 2219 Aluminum**  
**Tank Volume = 175 ft<sup>3</sup> (87 x 72.5 inch)**

**Tank Weight = 329.25 lbs**

**Tank Insulation – 34 layers of MLI**

## Chilldown Method:

- **6 Cycles of Charge-Hold-Vent Process**
- **Injection rates were measured**
- **714.35 lbs of LH2 was injected in 2.35 hrs**
- **Tank was filled to 94%**
- **Fluid and wall temperatures measured**
- **Estimated consumption of LH2 = 32 lbs**

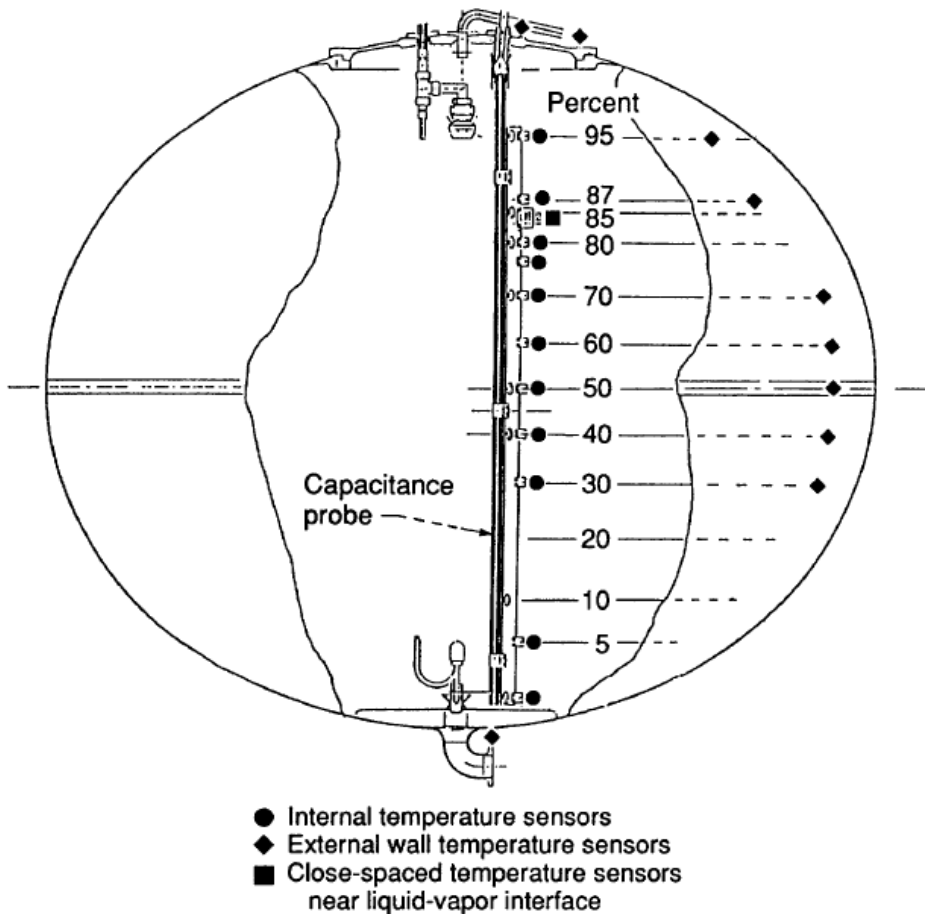
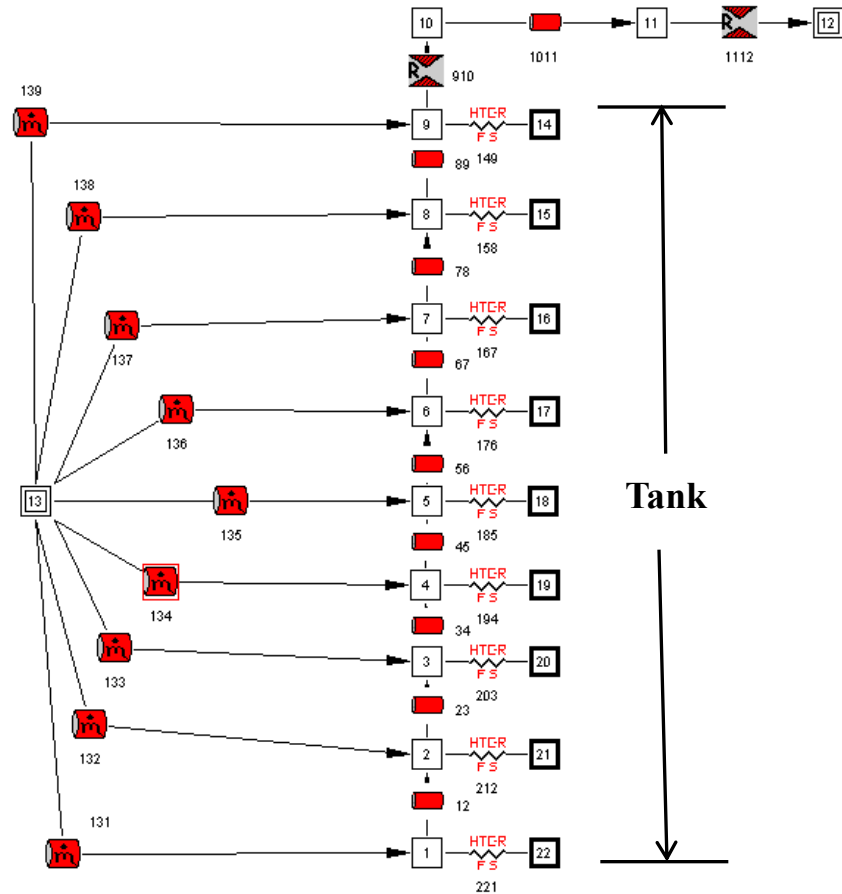
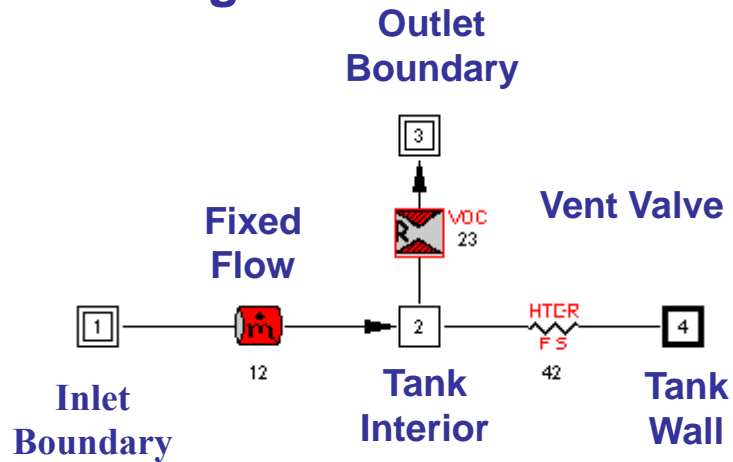


Figure 6.—Tank instrumentation.



## Nine Node Tank Model

### Single Node Tank Model



- No heat leak was assumed
- Initial Tank Pressure = 2 psia
- Initial Tank Temperature = -20 ° F

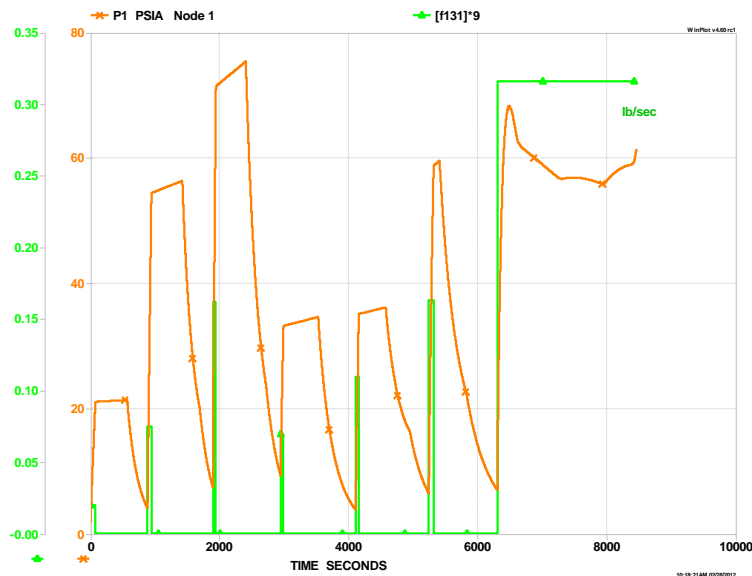




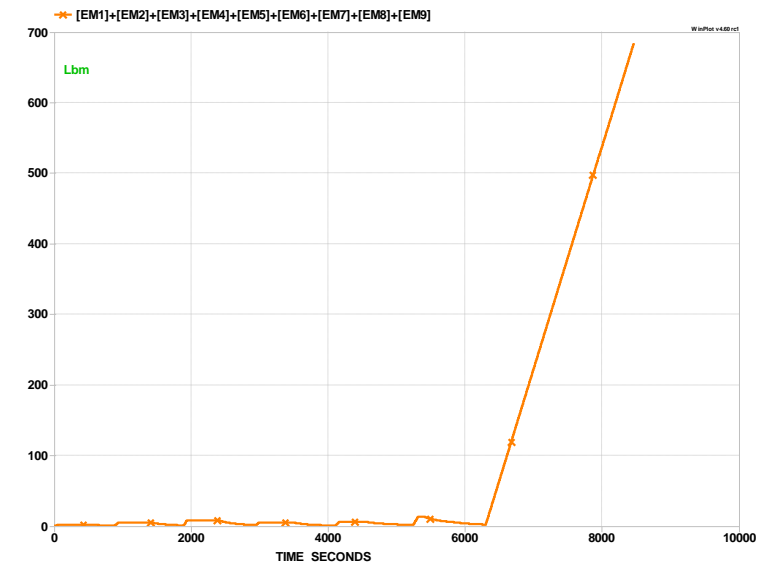
# GFSSP Model Results



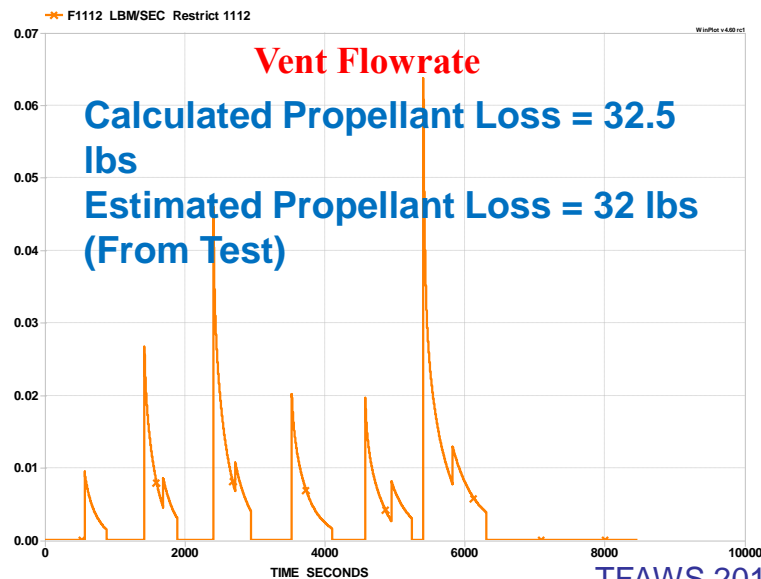
## Pressure & Inlet Flowrate



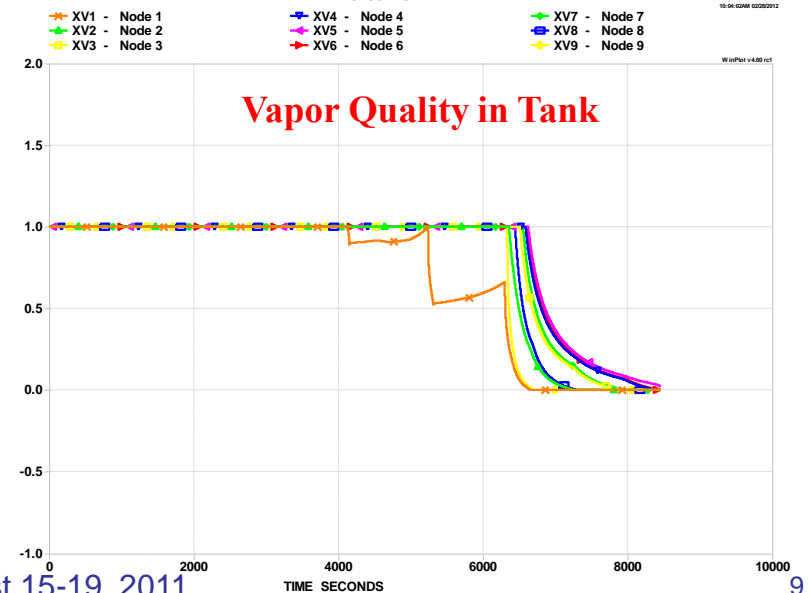
## LH2 Mass in Tank



## Vent Flowrate

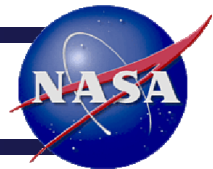


## Vapor Quality in Tank

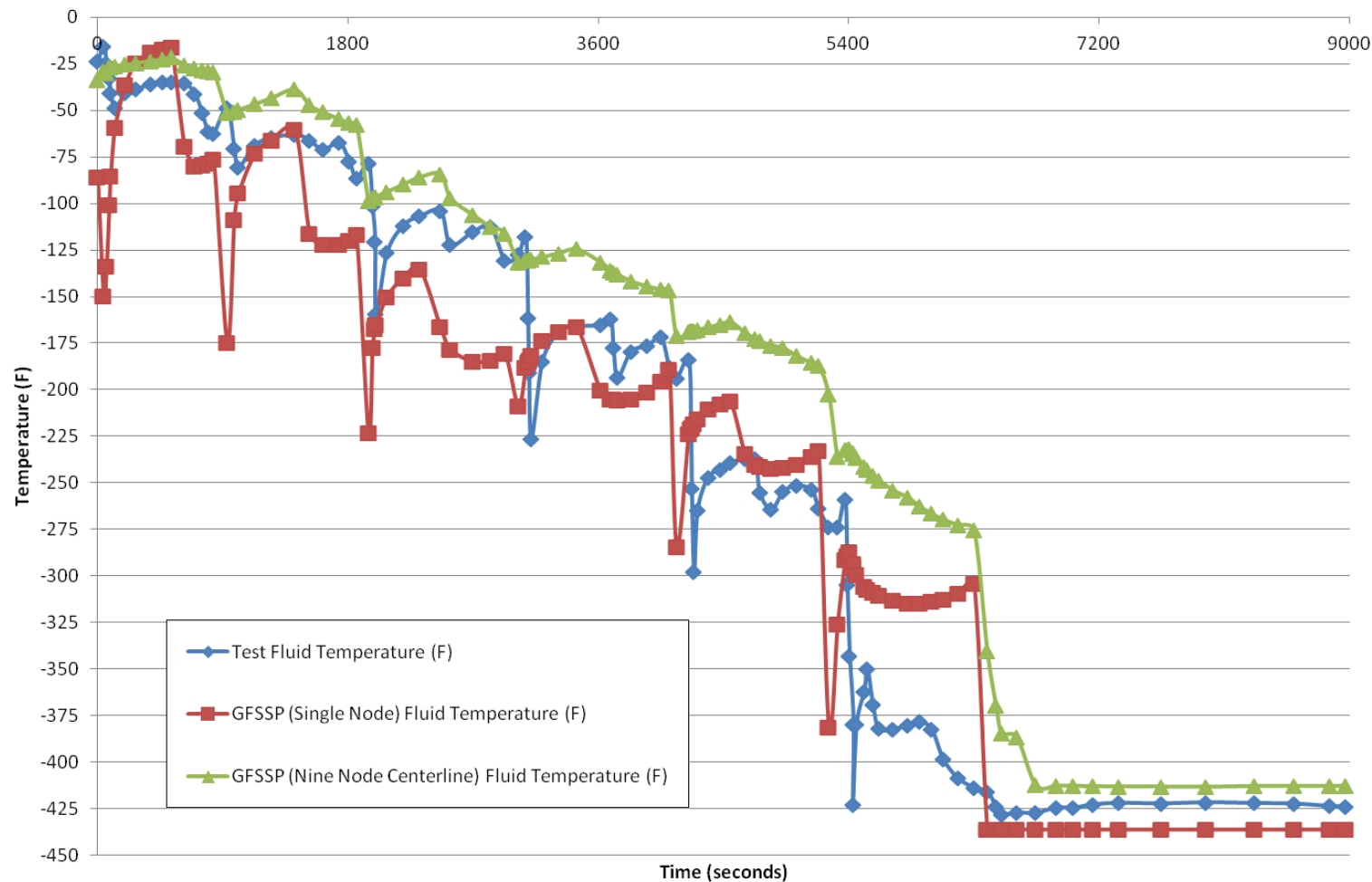


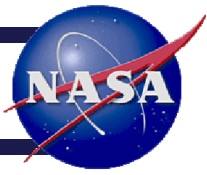


# COMPARISON WITH TEST DATA



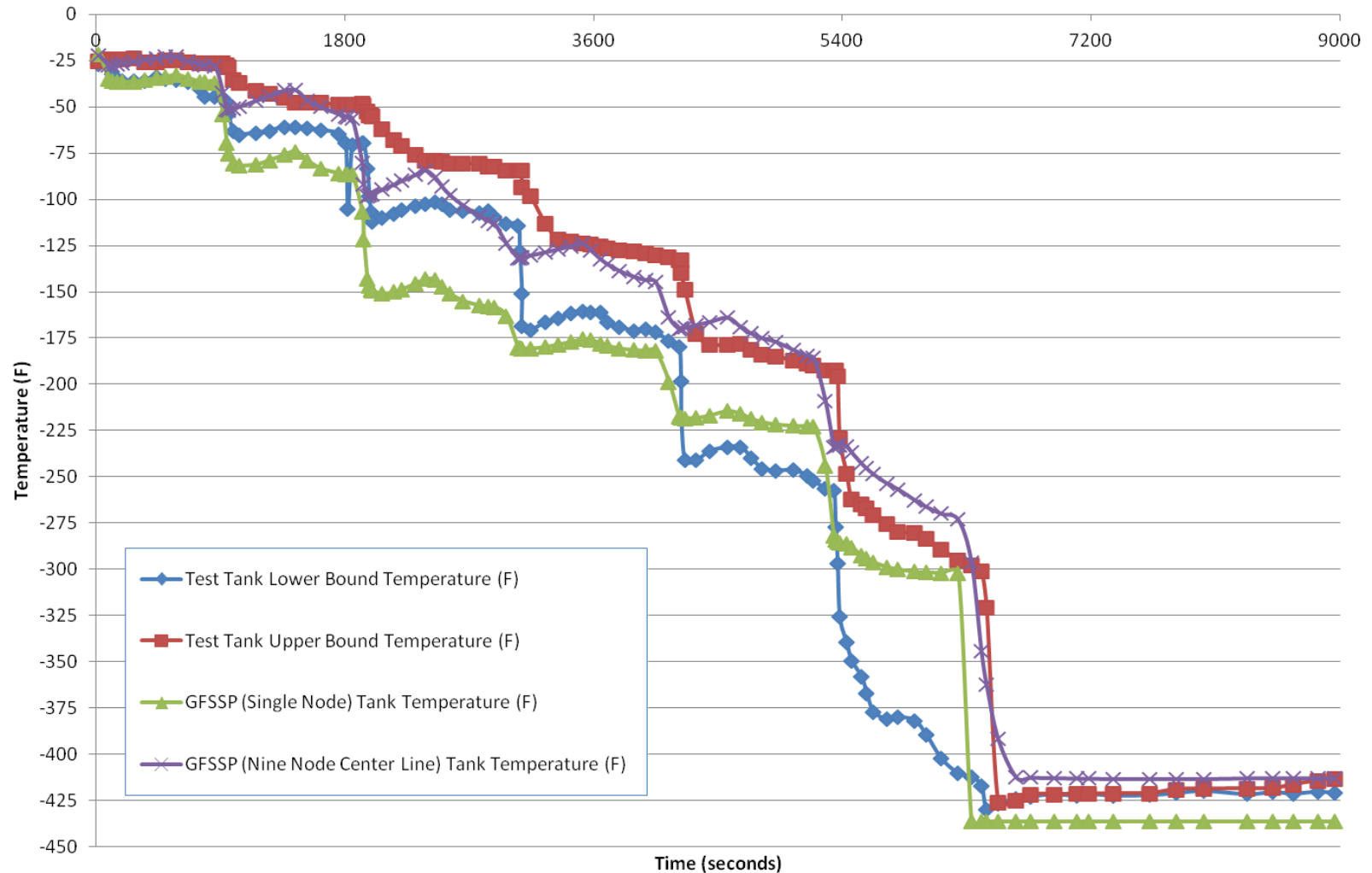
## GFSSP (Single Node) Fluid and GFSSP (9 Node Centerline) Fluid Temperature Results Comparison to Test Fluid Temperature Results





# COMPARISON WITH TEST DATA

## GFSSP (Single Node) and GFSSP (9 Node Centerline) Wall Temperature Results Comparison to Test Wall Temperature Results





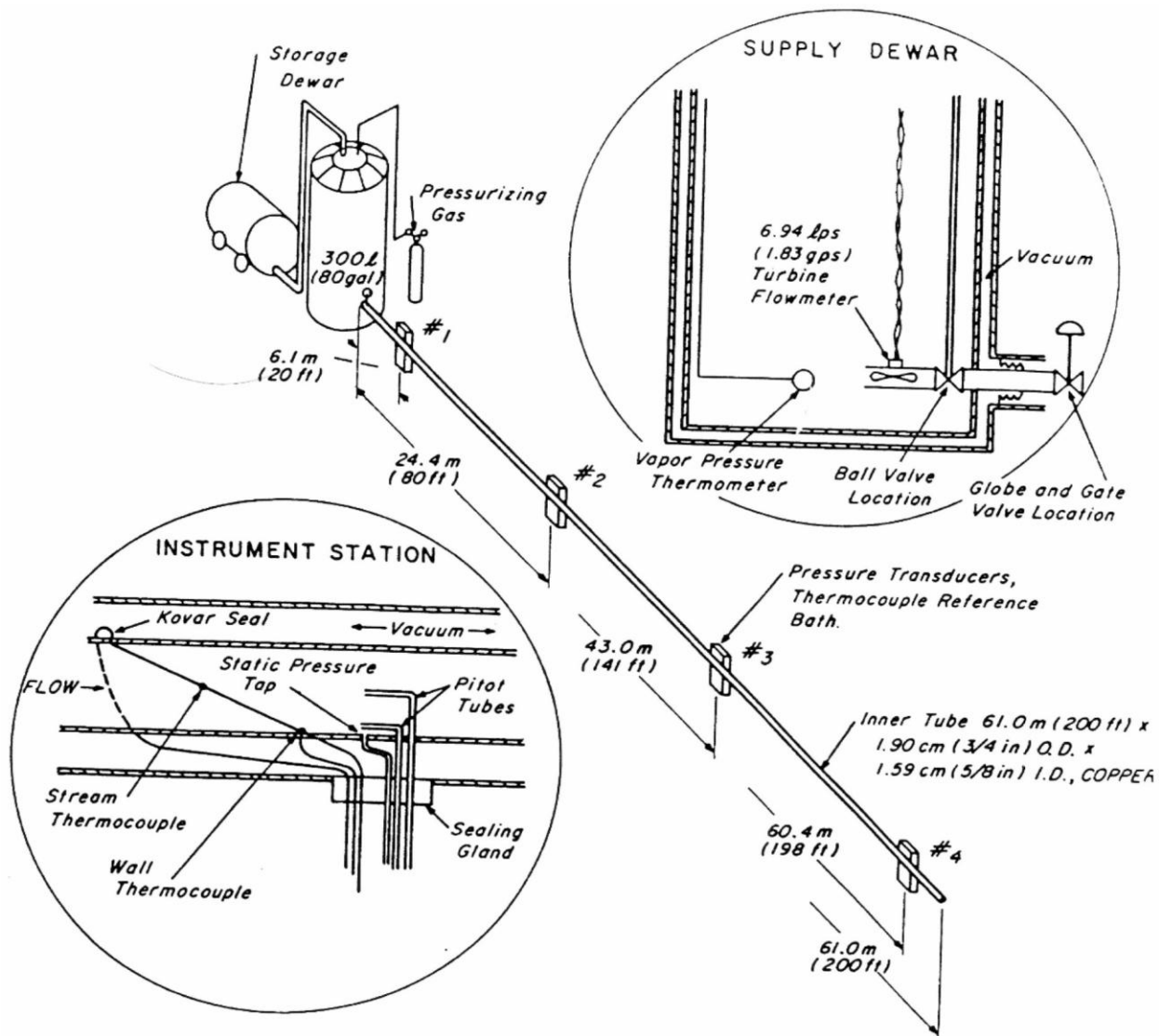
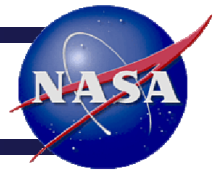
# Discussion of Results



- Predicted propellant loss agrees extremely well with estimated propellant loss during the test:
  - Predicted – 32.5 lbs (9-node model) & 33.5 lbs (1-node model)
  - Test – 32 lbs
- Pressure Predictions, however, are inaccurate due to use of fixed flowrate boundary condition at inlet
  - GFSSP's preferable option is to employ pressure boundary condition and calculate flowrate
  - K-site test report does not have enough information to model transfer line upstream of the tank where pressure is measured
- Future validation effort should be directed towards the development of integrated transfer line and tank model to simulate the entire chilldown process which includes both transfer line and tank
- Improved pressure prediction will also improve temperature prediction

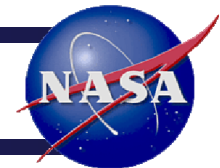


# NBS Test Set-up of Cryogenic Transfer Line





# GFSSP Model of Cryogenic Transfer Line



## Saturated LH<sub>2</sub> chilldown time for various driving pressures

Driving Pressure (psia)	Saturation Temperature (°F)	Experimental Chilldown Time (s)	Predicted Chilldown Time (s)
74.97	-411.06	68	70
86.73	-409.08	62	69
111.72	-406.4	42	50
161.72	-402.13	30	33

## Subcooled LH<sub>2</sub> chilldown time for various driving pressures. LH<sub>2</sub> is subcooled at -424.57 °F

Driving Pressure (psia)	Experimental Chilldown Time (s)	Predicted Chilldown Time (s)
36.75	148	150
61.74	75	80
86.73	62	60
111.72	41	45
136.72	32	35
161.7	28	30

## Saturated LN<sub>2</sub> chilldown time for various driving pressures

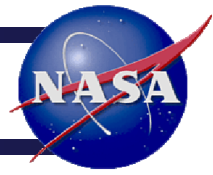
Driving Pressure (psia)	Saturation Temperature (°F)	Experimental Chilldown Time (s)	Predicted Chilldown Time (s)
61.74	-294.09	165	185
74.97	-289.71	150	160
86.73	-286.24	130	140

## Subcooled LN<sub>2</sub> chilldown time for various driving pressures. LN<sub>2</sub> is subcooled at -322.87 °F

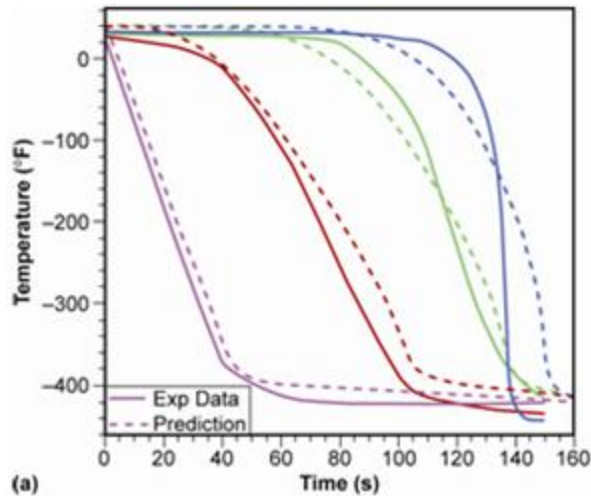
Driving Pressure (psia)	Experimental Chilldown Time (s)	Predicted Chilldown Time (s)
36.75	222	250
49.97	170	175
61.74	129	140
74.97	100	100
86.73	85	90



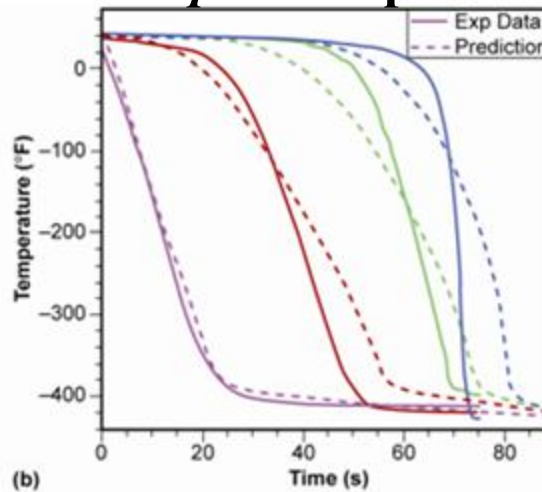
# Comparison of temperature histories for subcooled LH<sub>2</sub>



$p=36.74$  psia



$p=61.72$  psia

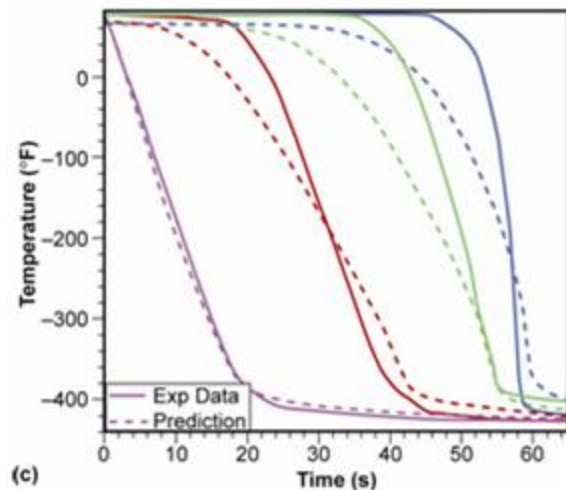


**Station #1 (violet) —  
20 ft from tank inlet**

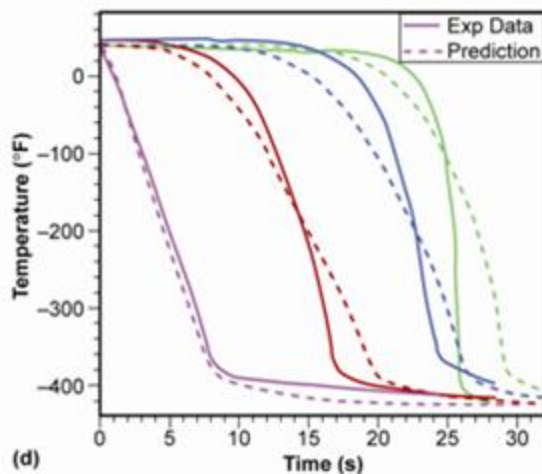
**Station #2 (red) —  
80 ft from tank inlet**

**Station #3 (green) —  
141 ft from tank inlet**

**Station #4 (blue) —  
198 ft from tank inlet**



$p=86.7$  psia



$p=161$  psia





## Discussion of Transfer Line Chillardown Results

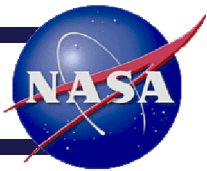


- Good agreement between test and prediction has been observed for chillardown time for different test conditions
- The model correctly predicts the effects of varying the inlet driving pressure on chillardown time for both subcooled and saturated conditions
- There is, however, discrepancy in temperature history between test and predictions
- The observed discrepancy may be attributed to the inaccuracy in heat transfer coefficient correlation
- An implementation of complete boiling curve that include the regimes of film boiling, transitional region and nucleate boiling may reduce or eliminate the observed discrepancies





# CONCLUDING REMARKS



- GFSSP is a general-purpose finite-volume based multi-node (flow network) code for steady and time-dependent flows, including modeling phase changes, conjugate heat transfer, compressibility, mixture thermodynamics, and external body forces such as gravity and centrifugal.
- Twenty-one different resistance/source options are provided for modeling momentum sources or sinks in the branches.
- Two thermodynamic property programs (GASP/WASP and GASPAK) provide required thermodynamic and thermo-physical properties for thirty six fluids
- This paper presents an application and partial verification of no-vent fill modeling of cryogenic tank and transfer line chilldown
- More experimental data are needed for further verification and validation of the code to model these processes
- Boiling heat transfer correlations for all regimes need to be incorporated
- GFSSP is available free of cost for Government use from MSFC Tech Transfer Office after completing the necessary paperwork
- 3-day Training Class is offered at MSFC & KSC; a shorter version is offered at TFAWS