



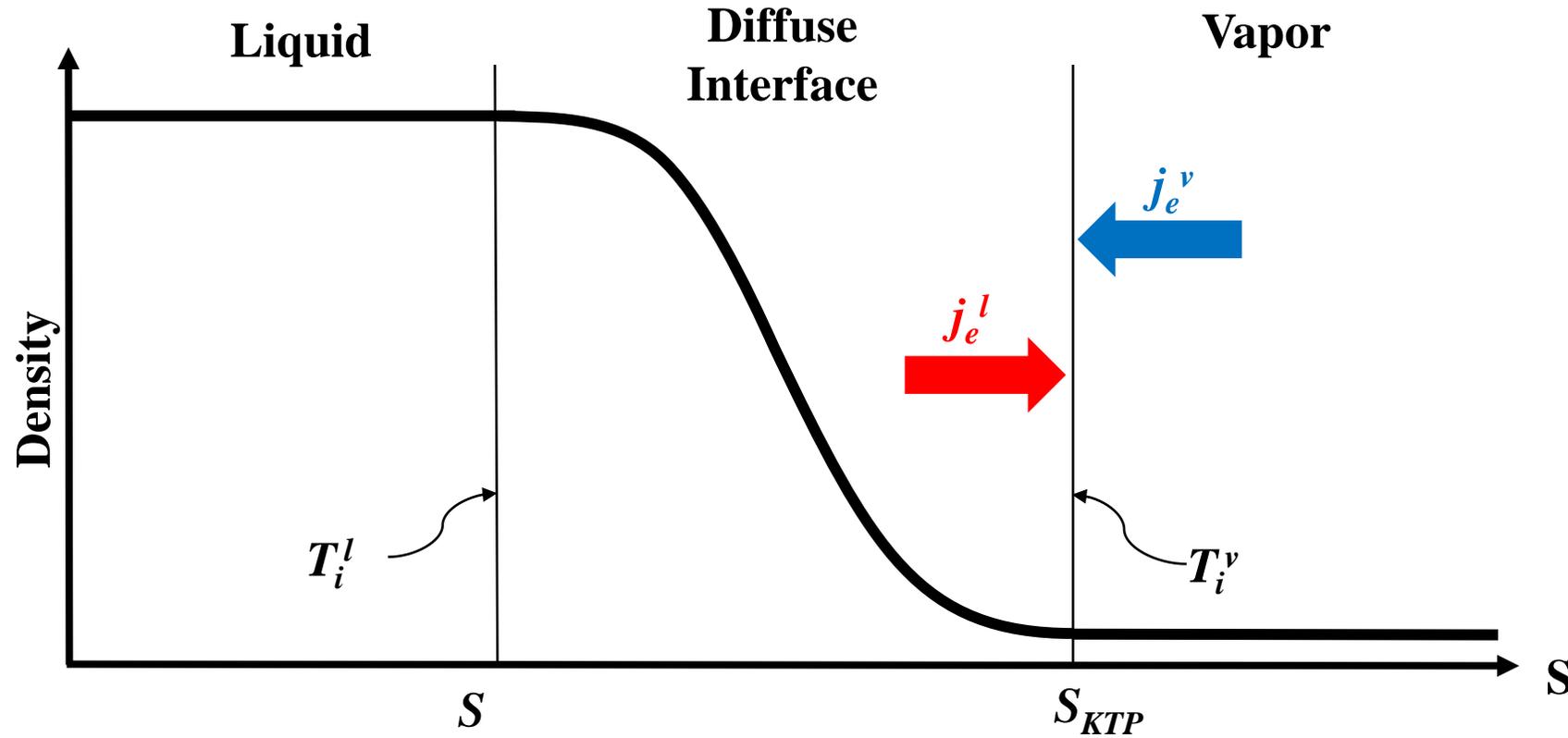
Michigan Tech



Testing simplified assumptions during liquid vapor phase change using data from International Space Station (ISS) experiments

Unmeelan Chakrabarti, Kishan Bellur, Jeffrey S Allen

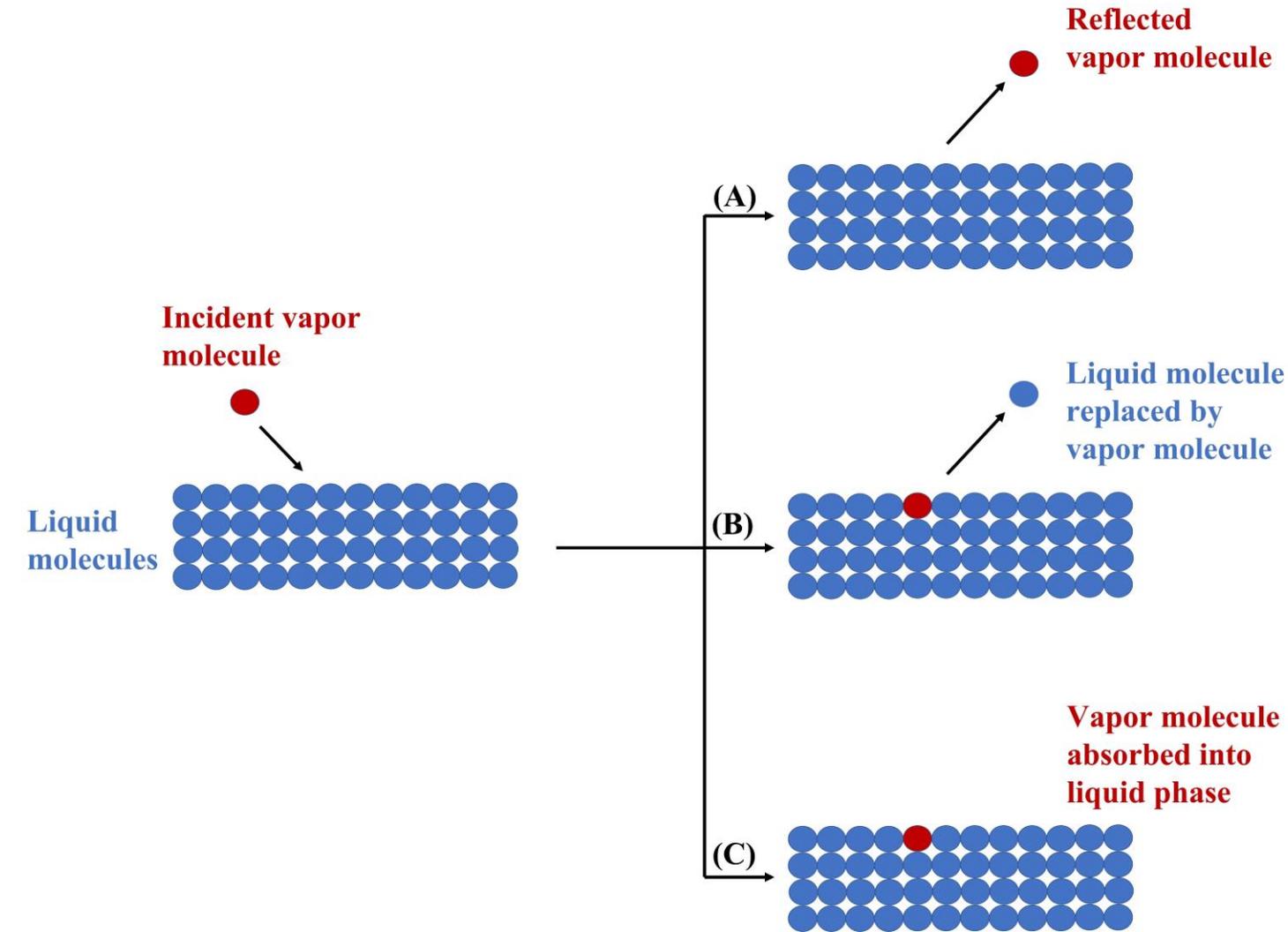
Kinetic theory and phase change



Phase change mass flux:

$$\dot{m}'' = \sqrt{\frac{k_B}{2\pi m}} \left[\underbrace{\rho_{sat}^v(T_i^l)}_{\text{evaporation}} \sqrt{T_i^l} - \underbrace{\rho_{sat}^v(T_i^l)}_{\text{condensation}} \sqrt{T_i^l} \right]$$

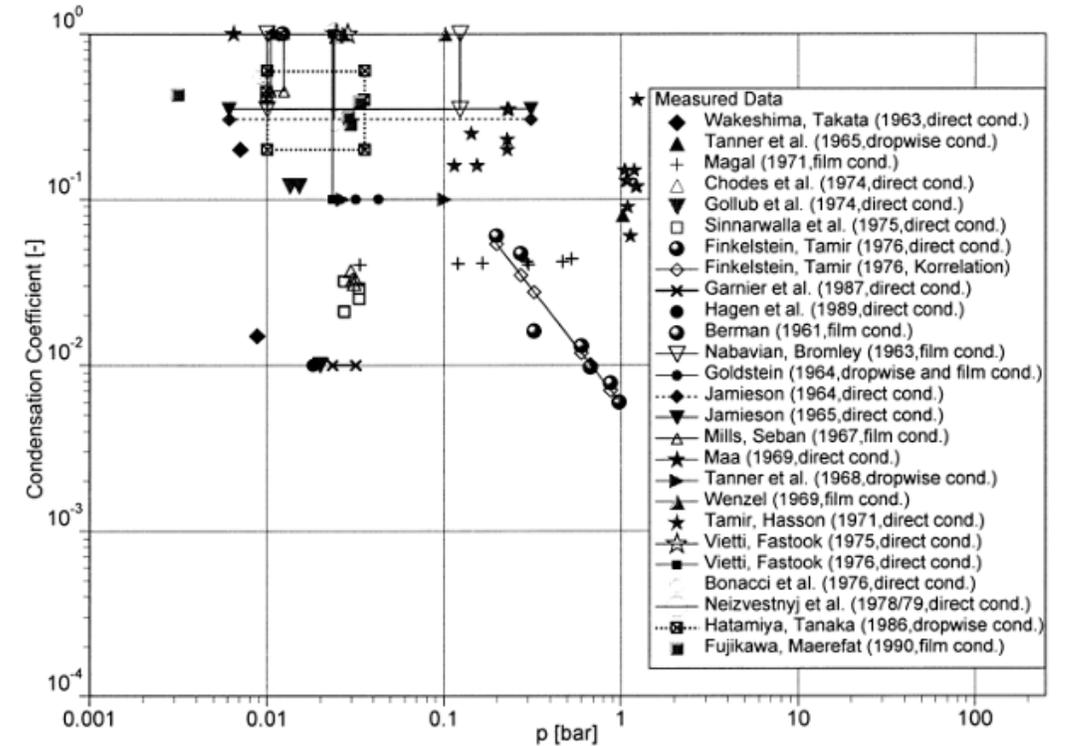
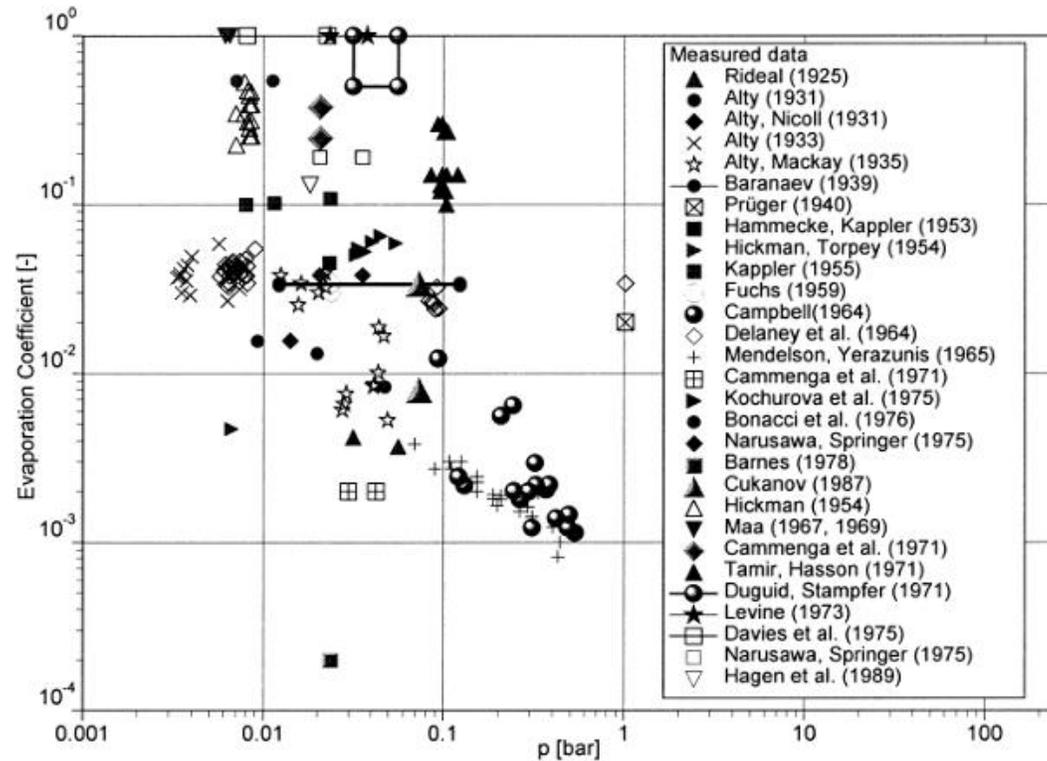
Kinetic theory and phase change



- Coefficients α_c and α_e are the fraction of molecules undergoing condensation and evaporation respectively.
- Phase change mass flux modified and rewritten in terms of saturation pressures using ideal gas laws:

$$\dot{m}'' = \left(\frac{2}{2 - \alpha_c} \right) \sqrt{\frac{m}{2\pi k_B}} \left[\alpha_e \frac{P_s(T_i^l)}{\sqrt{T_i^l}} - \alpha_e \frac{P^v}{\sqrt{T_i^v}} \right]$$

Kinetic theory and phase change

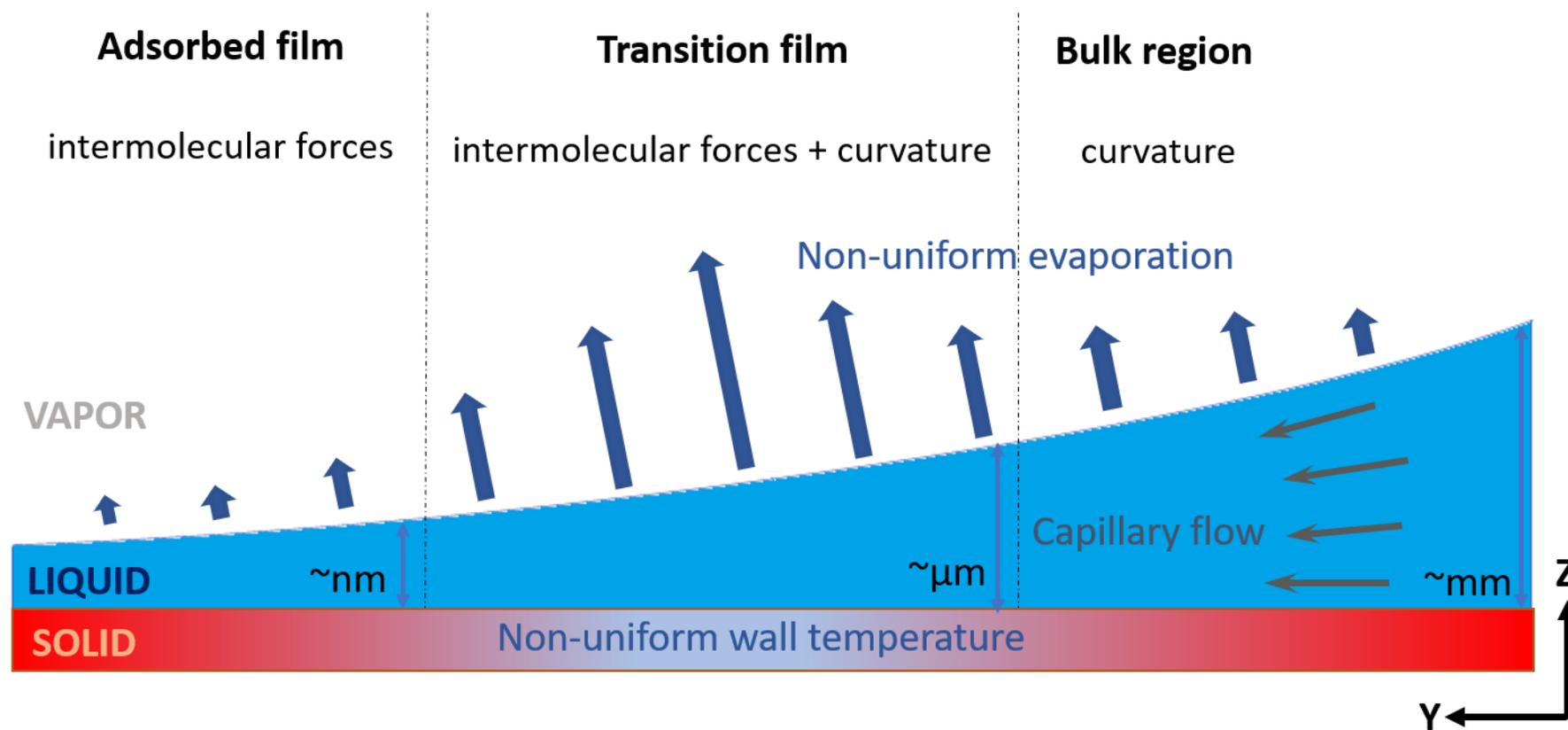


Simplified assumption: Equal evaporation and condensation coefficients ($\alpha_e = \alpha_c$)?

[1] Marek, R. et al. International Journal of Heat and Mass Transfer. 2001

[2] Robert . W. Schrage, PhD Dissertation. 1953

Modified equation for curved interfaces



$$\dot{m}'' = \left(\frac{2\alpha_c}{2 - \alpha_c} \right) \frac{P_i^v}{\pi v_{mp}} \left[\beta W \sqrt{\frac{T_i^v}{T_i^l}} - 1 \right]$$

$$\beta = \frac{\alpha_e}{\alpha_c}$$

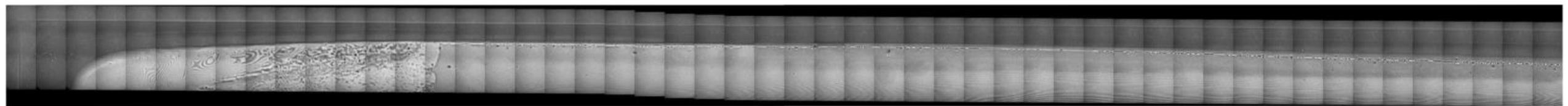
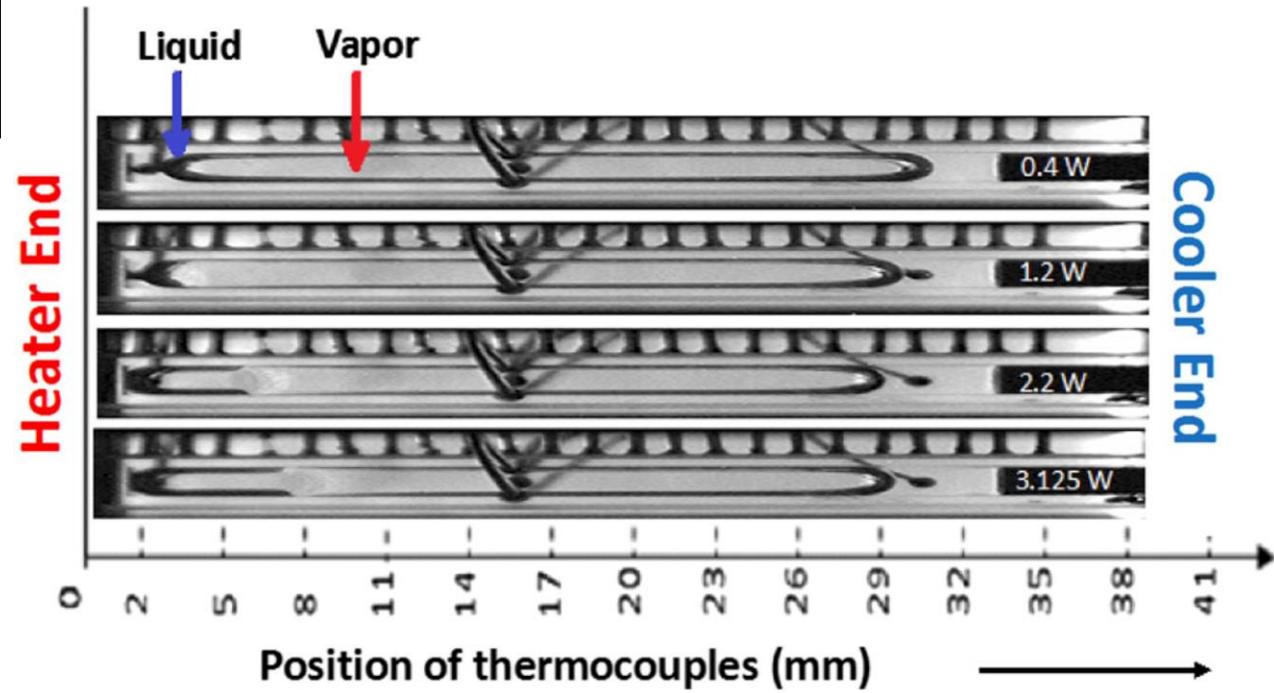
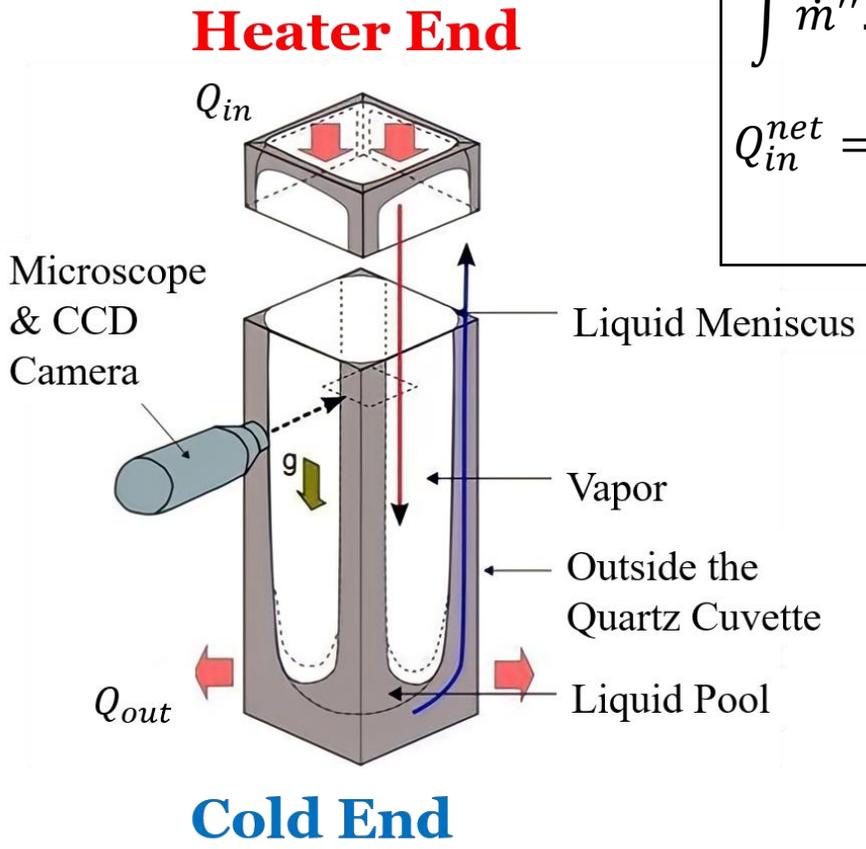
$$W = \frac{P_s(T_i^l)}{P_i^v} = \frac{P_{sat}^v}{P} + \frac{\rho_{sat}^v h_{fg}}{P_i^v} \left(1 - \frac{T_i^v}{T_i^l} \right) + \left(\frac{T_i^v}{T_i^l} \right) \left(\frac{\rho_{sat}^v}{\rho^l} \right) \left(\frac{\Pi + P_c}{P_i^v} \right)$$

Constrained Vapor Bubble (CVB) experiments

Unique constraints:

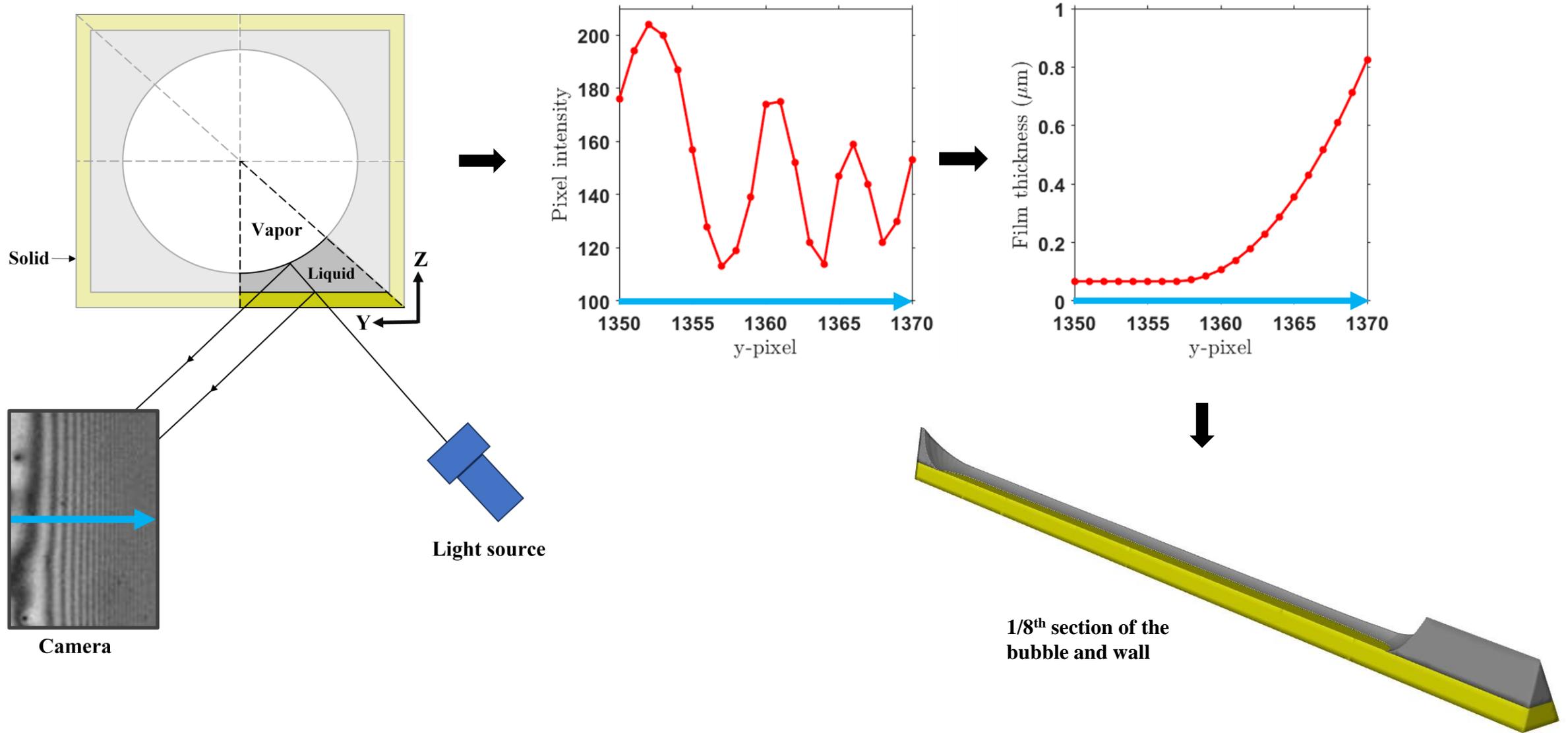
$$\int \dot{m}'' \cdot dA = 0$$

$$Q_{in}^{net} = hfg \int_{evap} \dot{m}'' \cdot dA$$

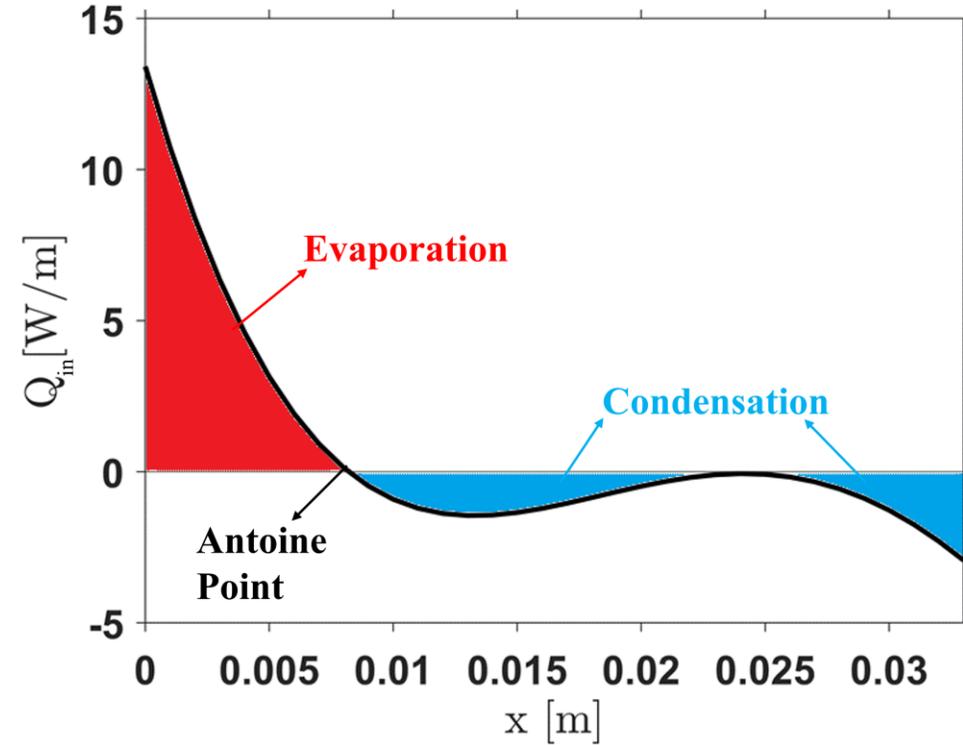
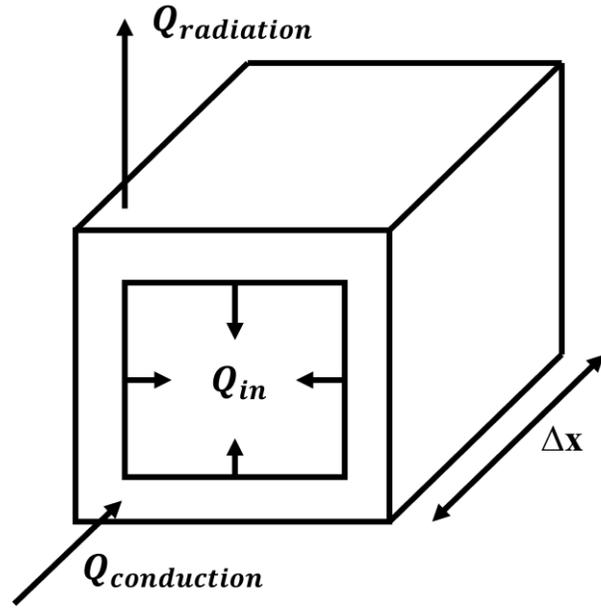


[4] P. C. Wayner et. al. Journal of Thermophysics and Heat Transfer. 2013

Interface Reconstruction



Heat balance



$$Q_{in}^{net} = \int_{evap} Q_{in} \cdot dx = \int_{evap} \left[\sigma P_o \epsilon (T(x)^4 - T_{\infty}^4) - KA_c \frac{d^2 T(x)}{dx^2} \right] \cdot dx$$

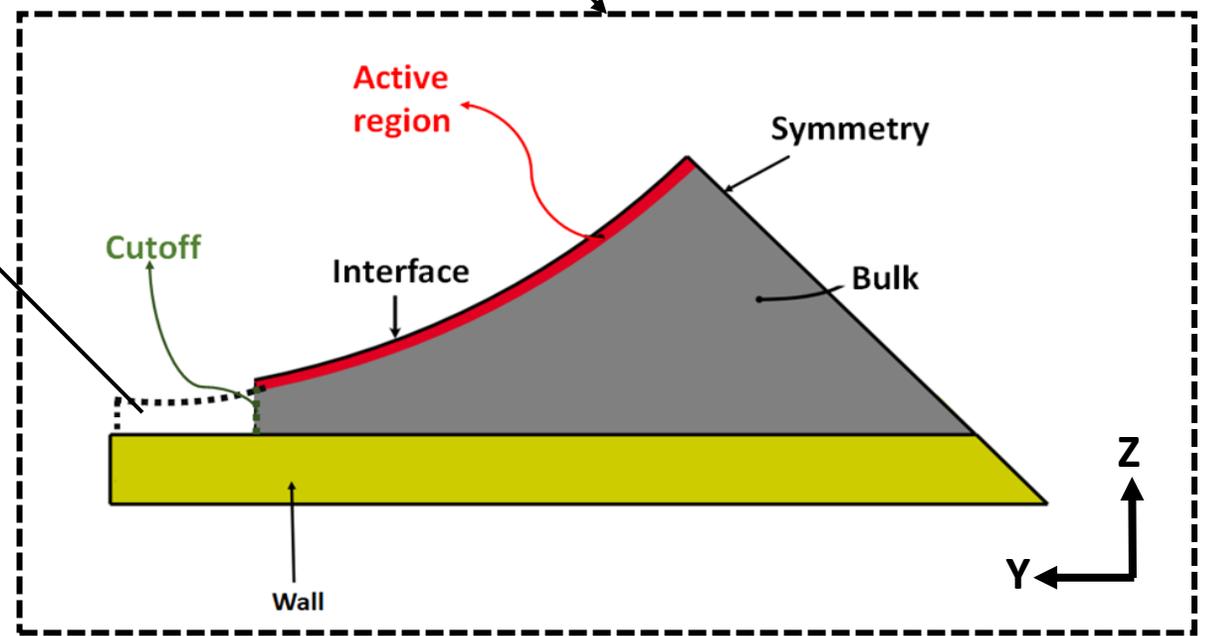
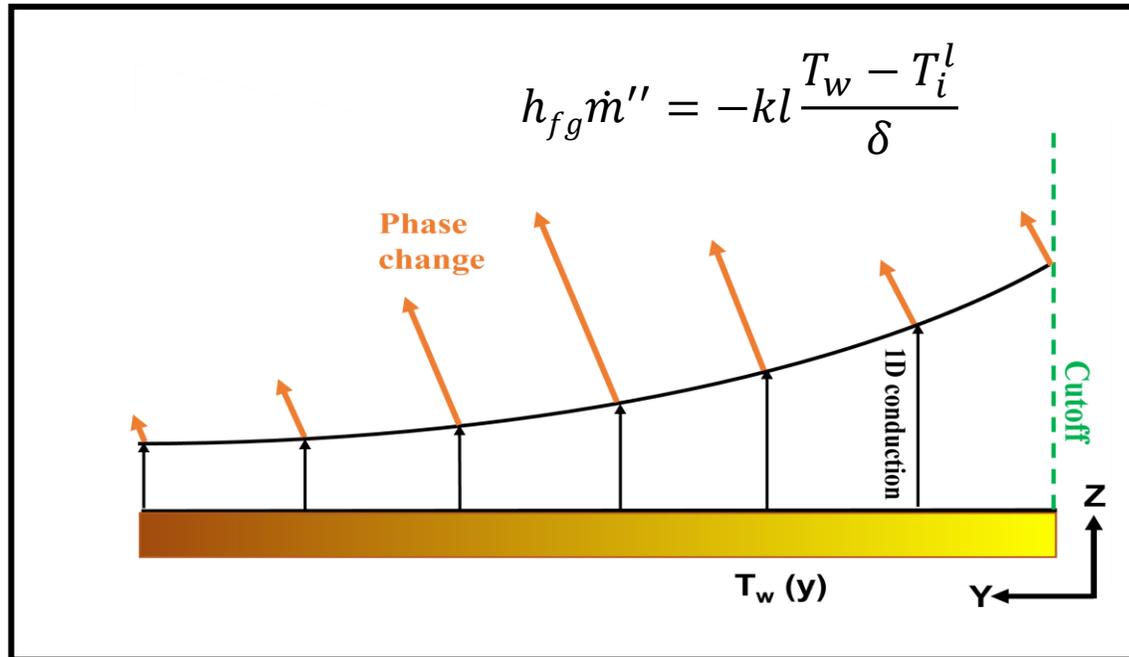
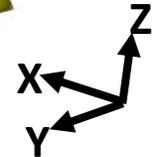
Multiscale model

Cold end

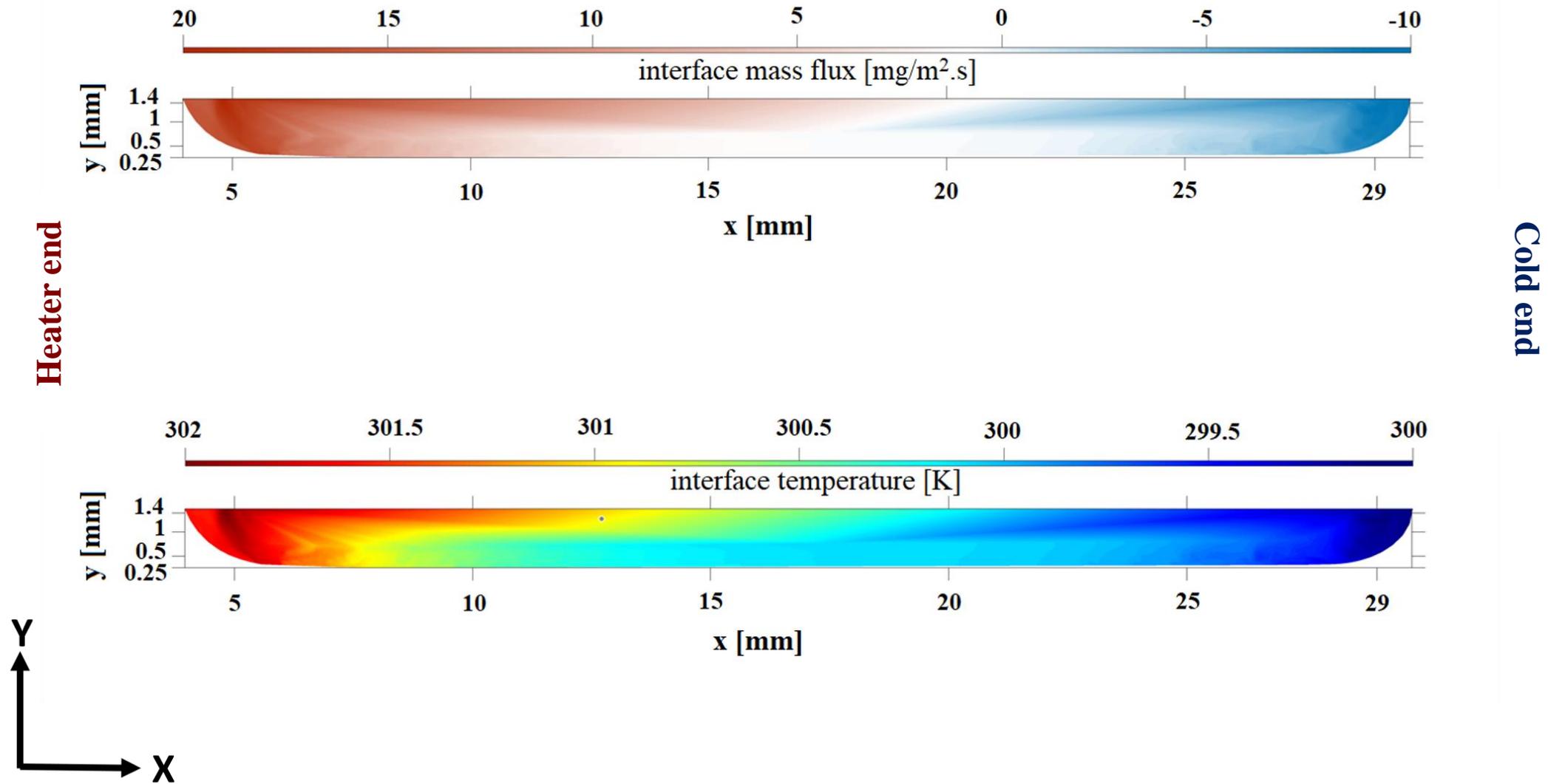
$$\dot{m}'' = \left(\frac{2\alpha_c}{2 - \alpha_c} \right) \frac{P_i^v}{\pi v_{mp}} \left[\beta W \sqrt{\frac{T_i^v}{T_i^l}} - 1 \right]$$

Heater end

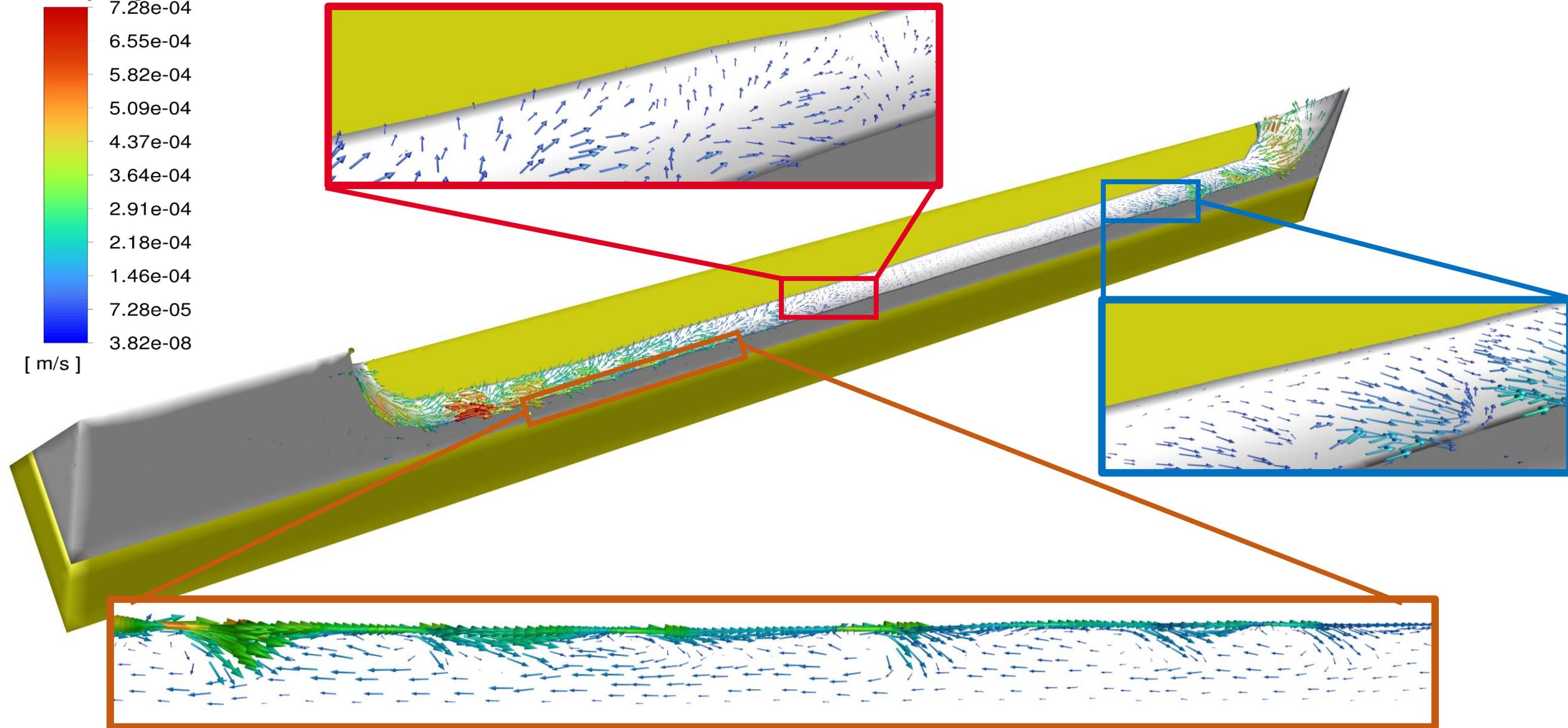
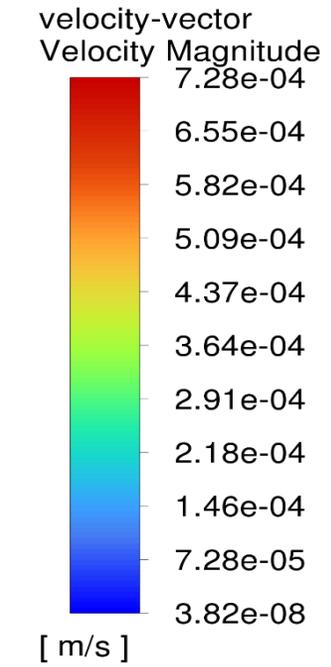
Liquid
Solid



Surface non-uniformities

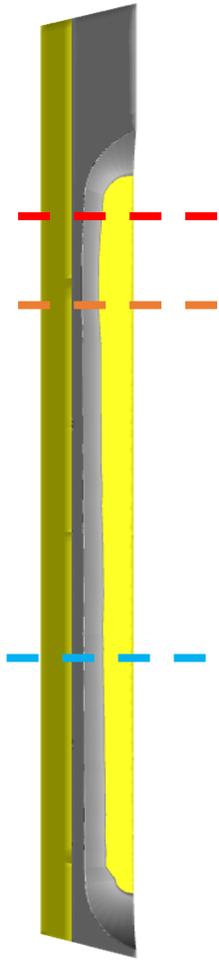
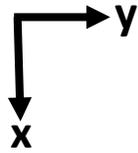


Surface non-uniformities



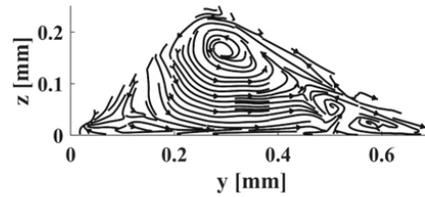
Surface non-uniformities

Heater end

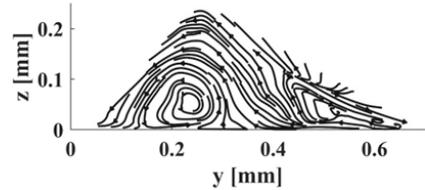


Cold end

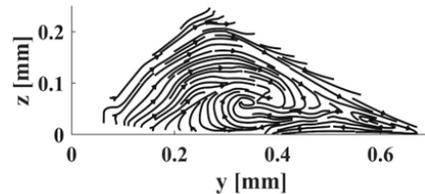
$x = 6\text{mm}$



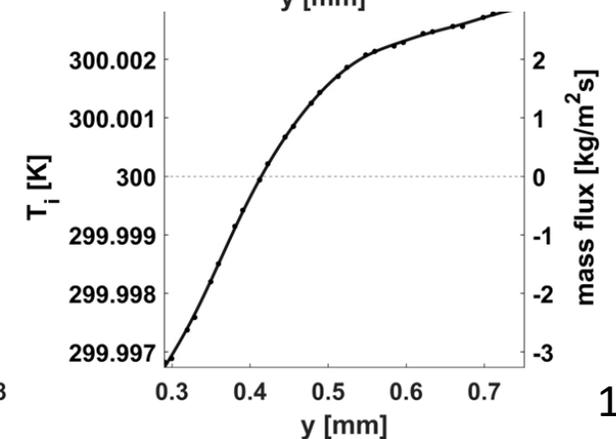
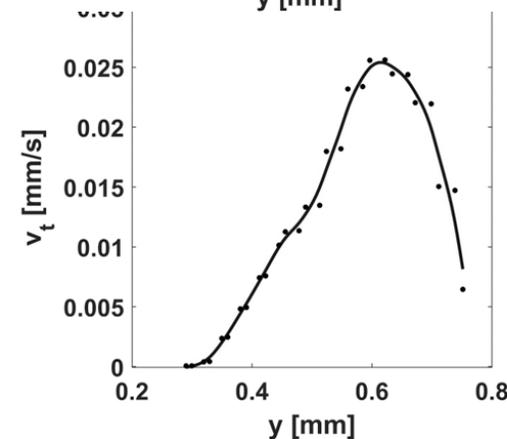
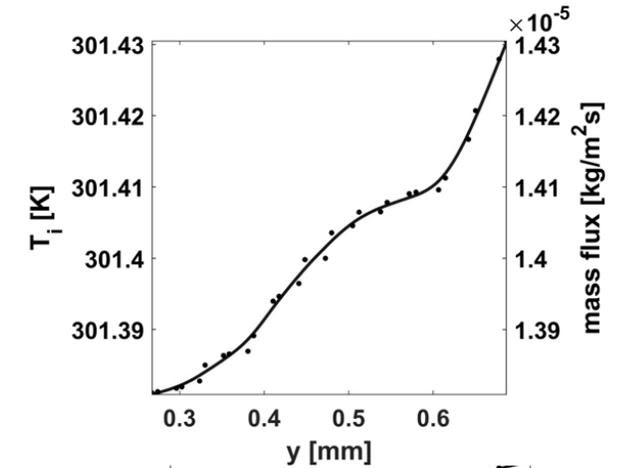
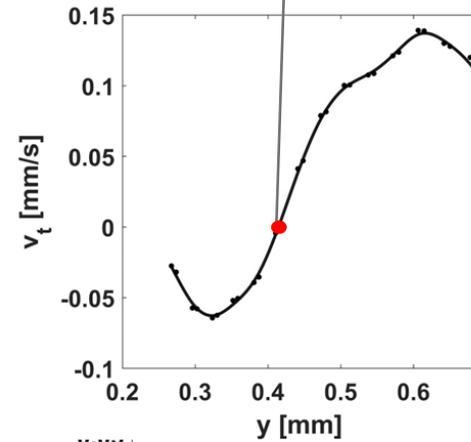
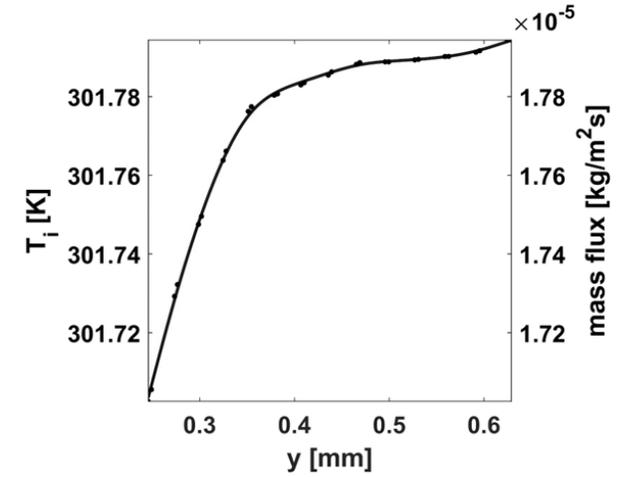
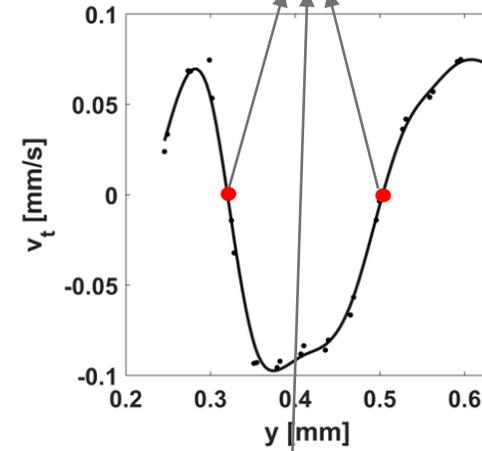
$x = 8\text{mm}$



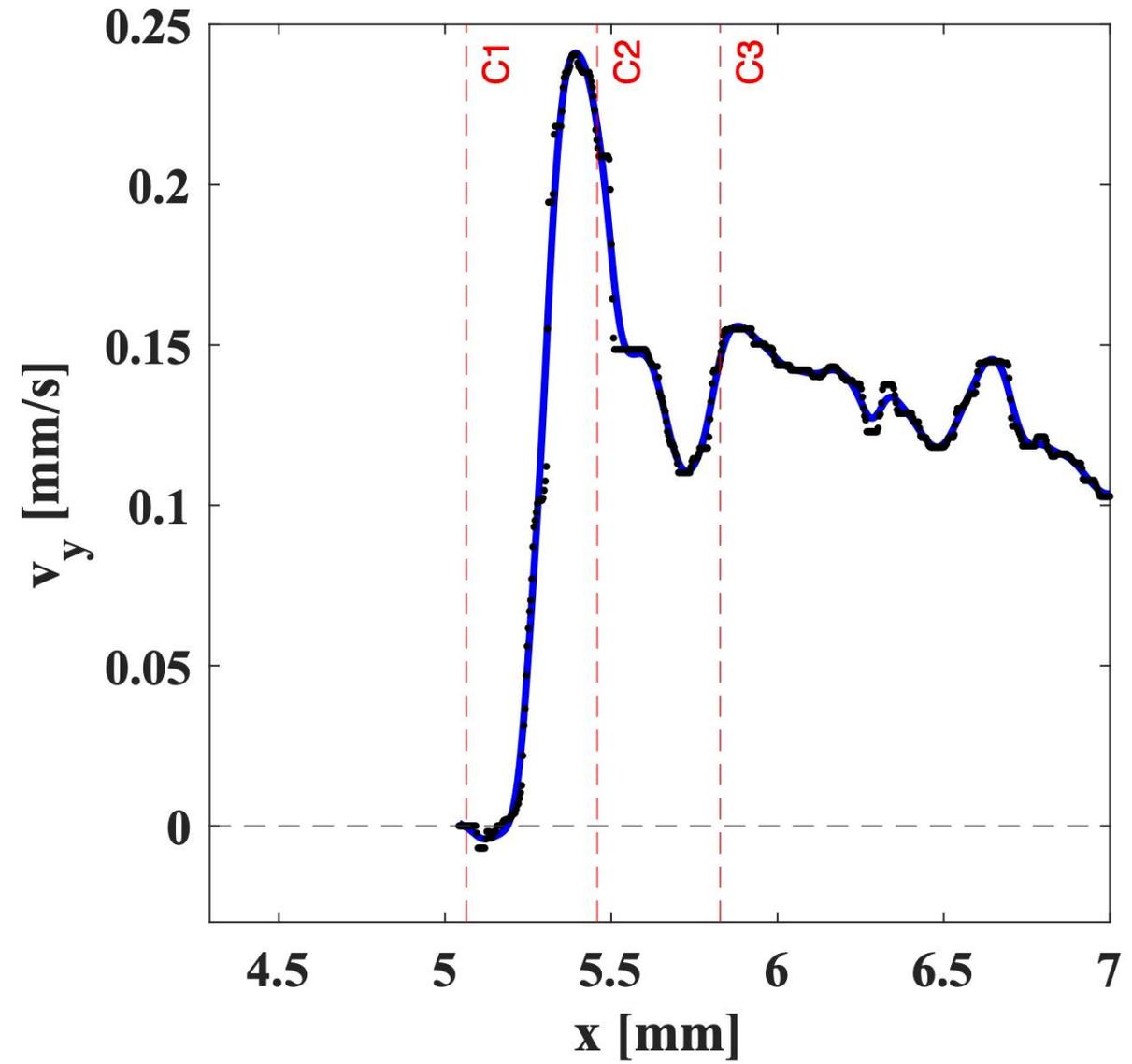
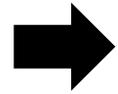
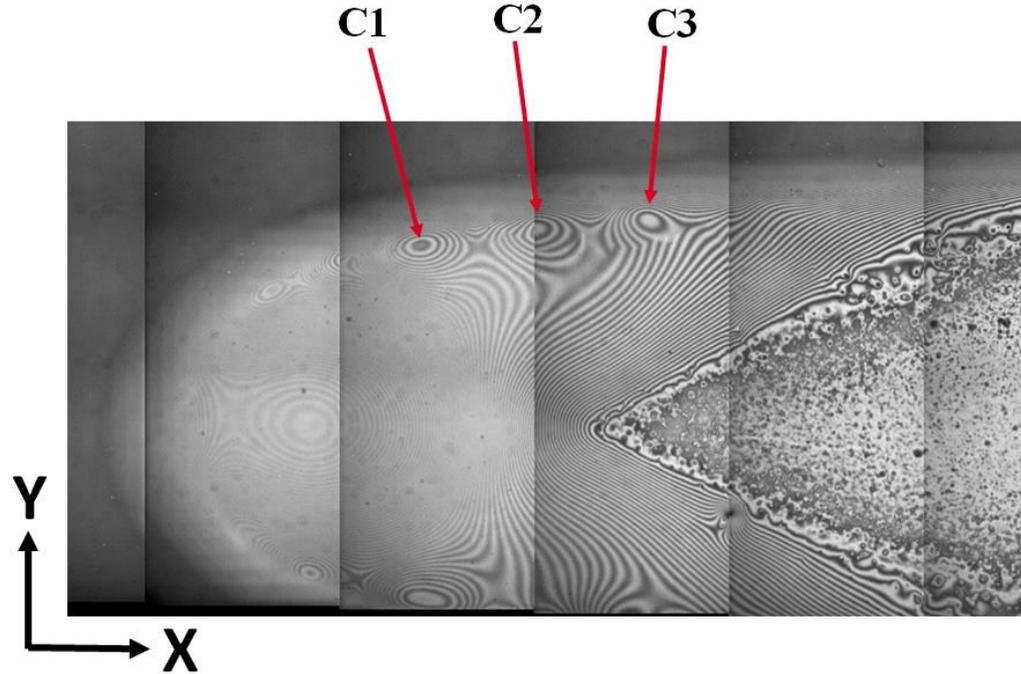
$x = 18\text{mm}$



Stagnation points

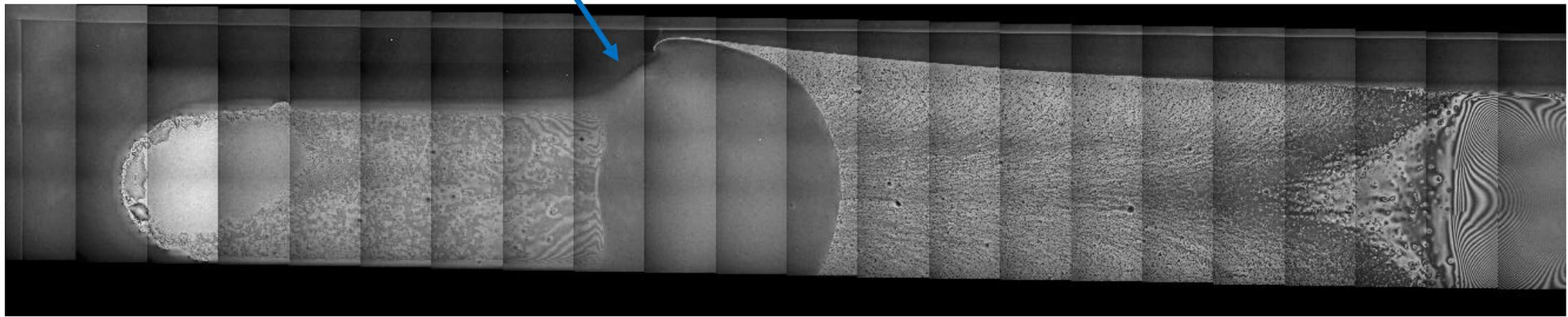
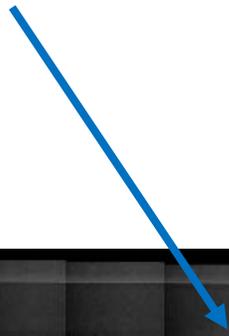


Surface non-uniformities



Surface non-uniformities

Liquid flooding near the heater



Heater side (3W)

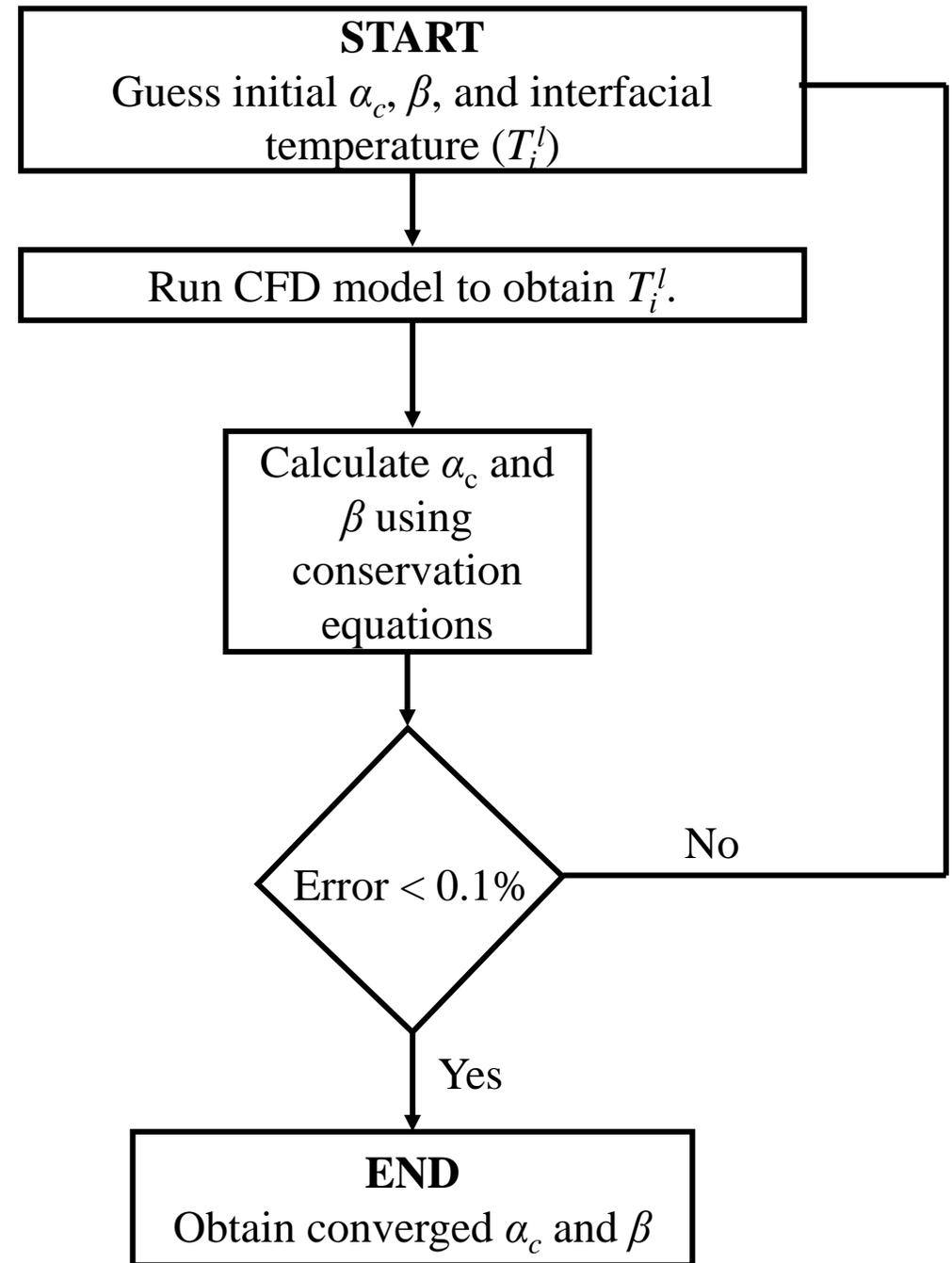
Calculation for α_c and β

$$\dot{m}'' = \left(\frac{2\alpha_c}{2 - \alpha_c} \right) \frac{P_i^v}{\pi v_{mp}} \left[\beta W \sqrt{\frac{T_i^v}{T_i^l}} - 1 \right]$$

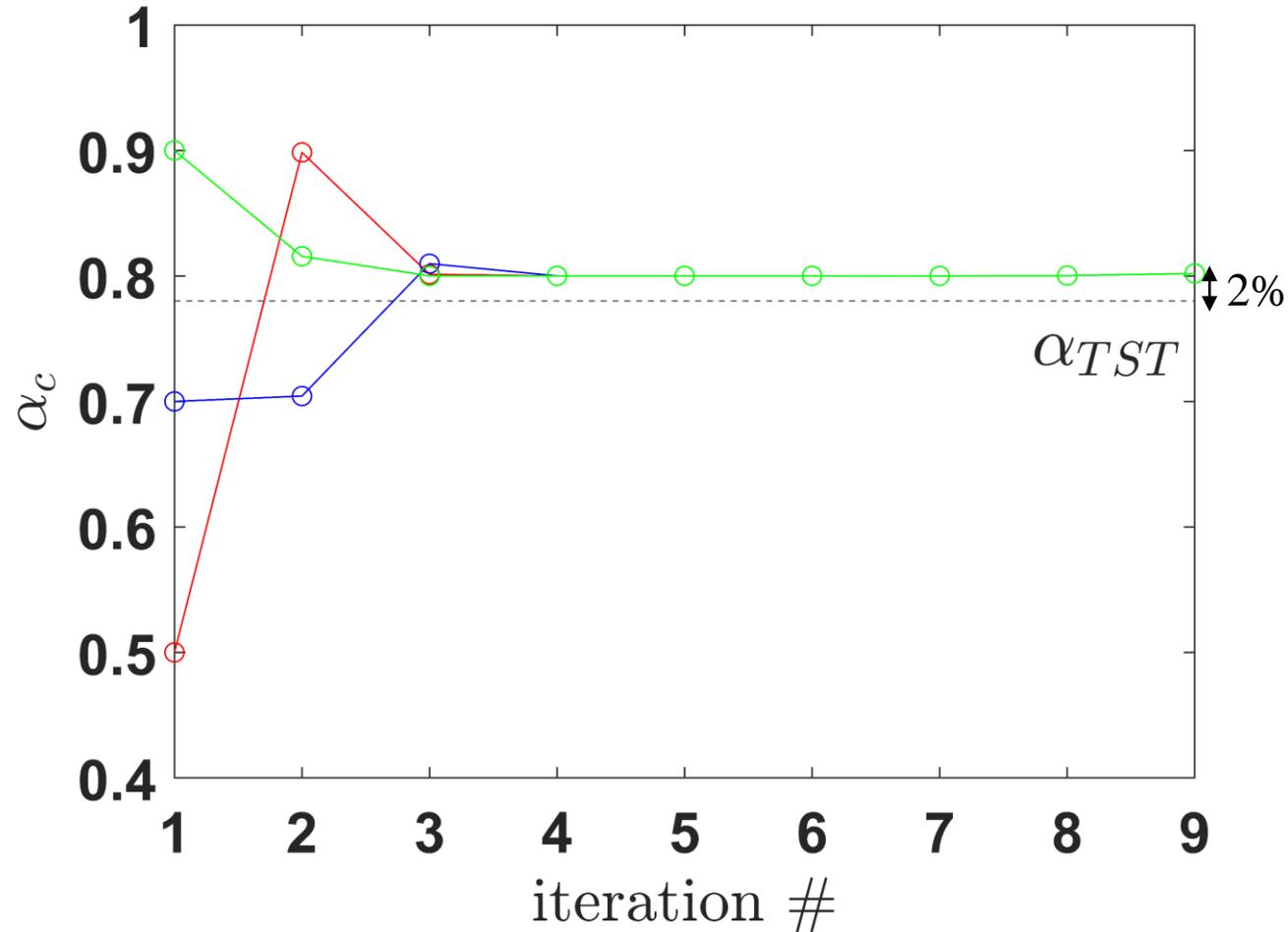
$$Q_{in}^{net} = hfg \int_{evap} \dot{m}'' \cdot dA$$

$$\beta = \frac{\alpha_e}{\alpha_c}$$

$$\int \dot{m}'' \cdot dA = 0$$



Results for α_c and β



$$\alpha_c = 0.8003, \beta = 0.9967,$$

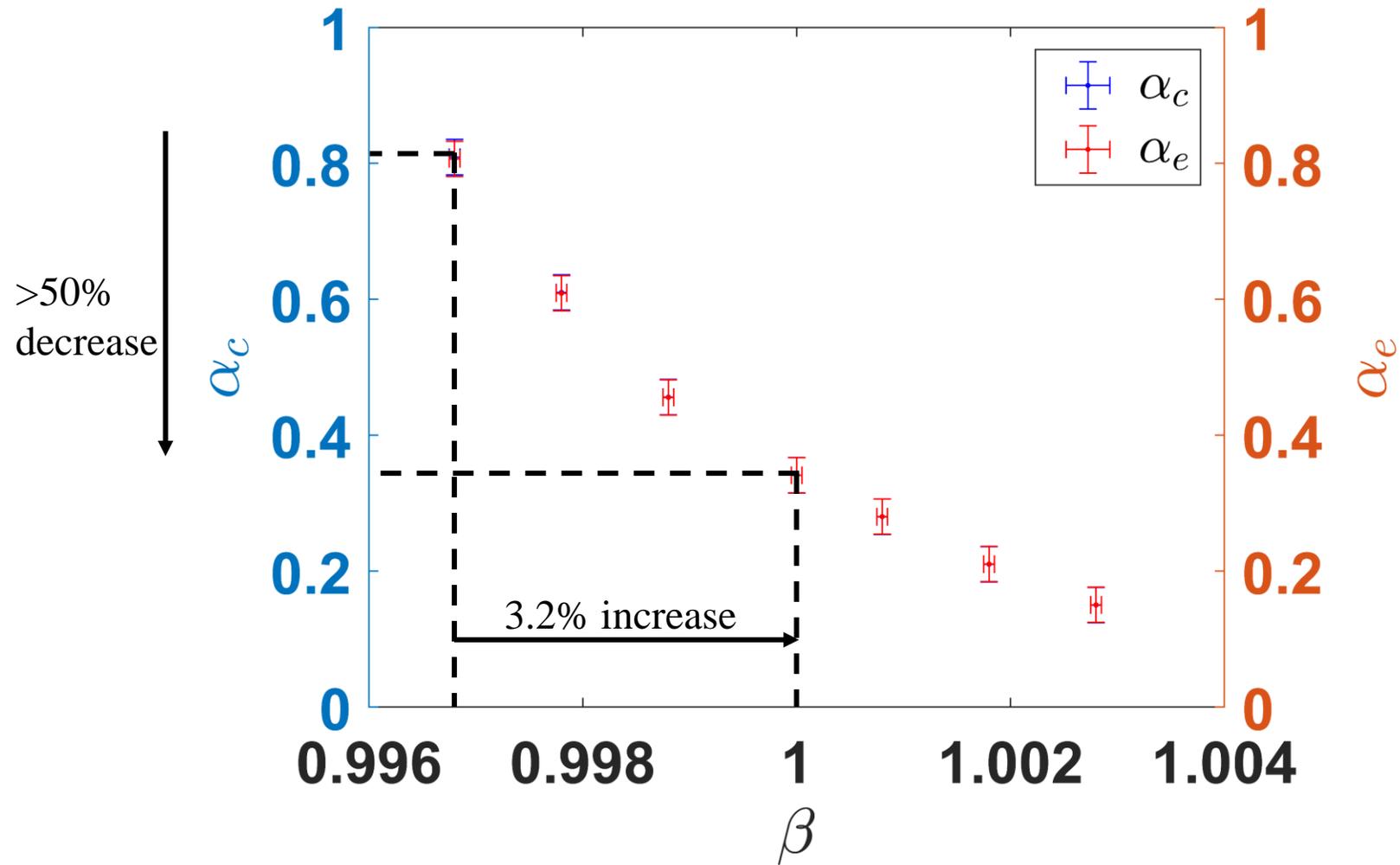
$$\alpha_e = \beta\alpha_c = 0.797$$

Average condensation coefficient
from Transition State Theory ^[4]:

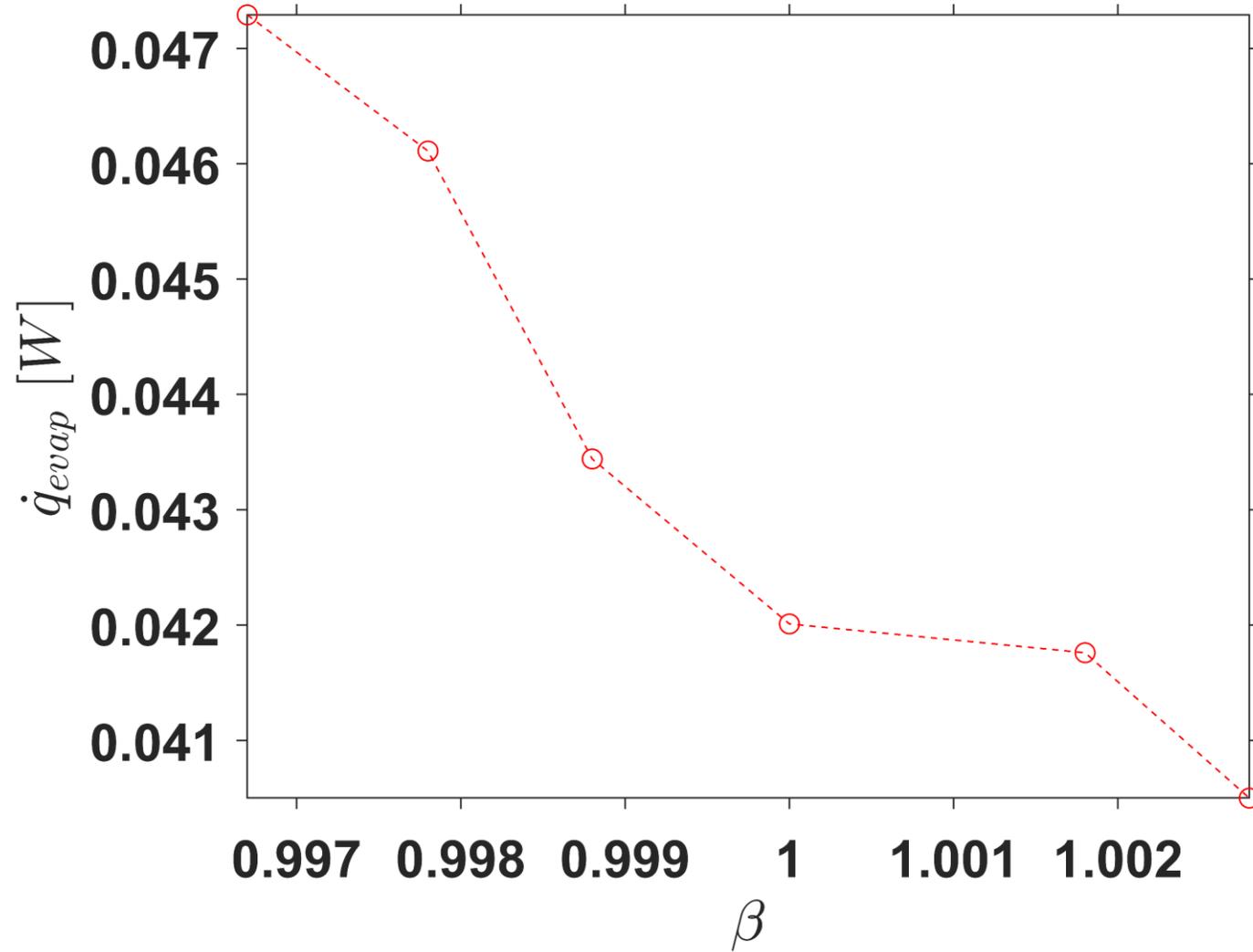
$$\alpha_{TST} = (1 - \sqrt[3]{l}) \exp\left(-\frac{1}{2} \frac{1}{\sqrt[3]{l} - 1}\right)$$

$$l = \frac{p_{sat}^v}{p_{sat}^l}$$

Results with fixing β

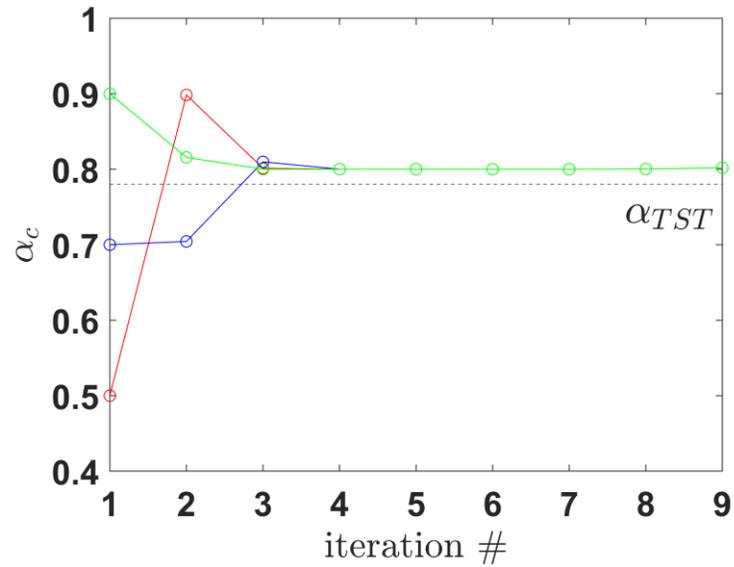


Results with fixing β

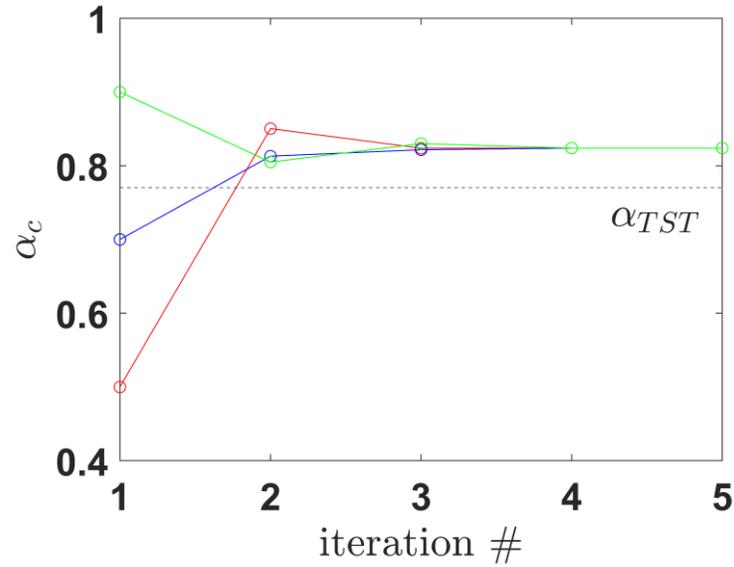


Varying heater settings

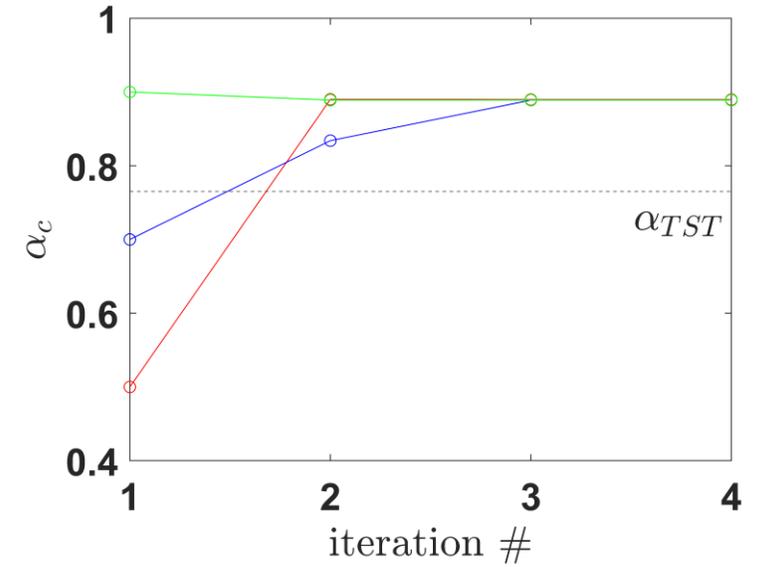
0.2 W setting



0.4 W setting



0.6 W setting



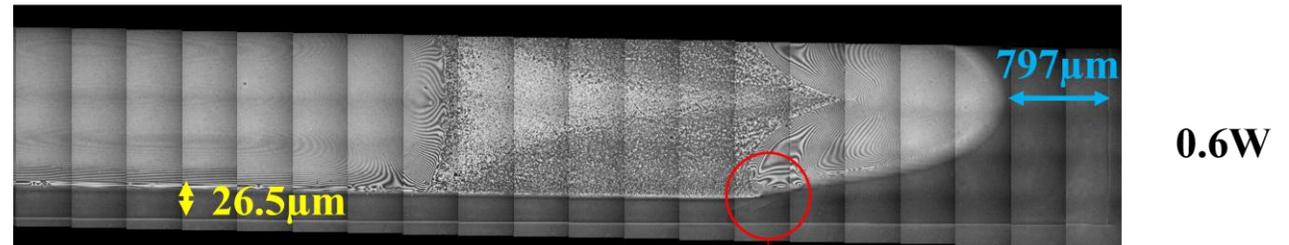
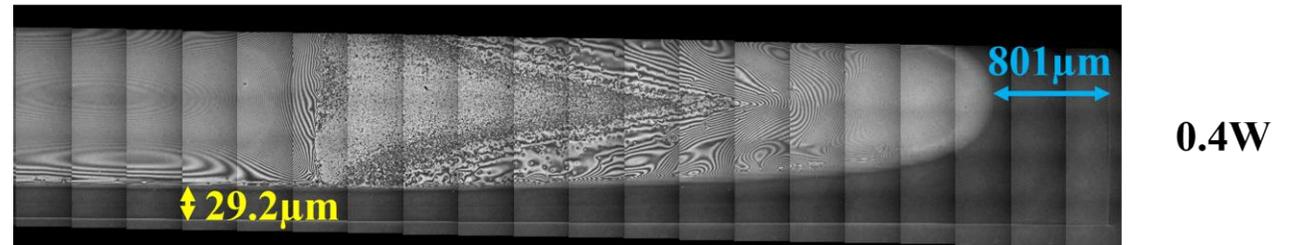
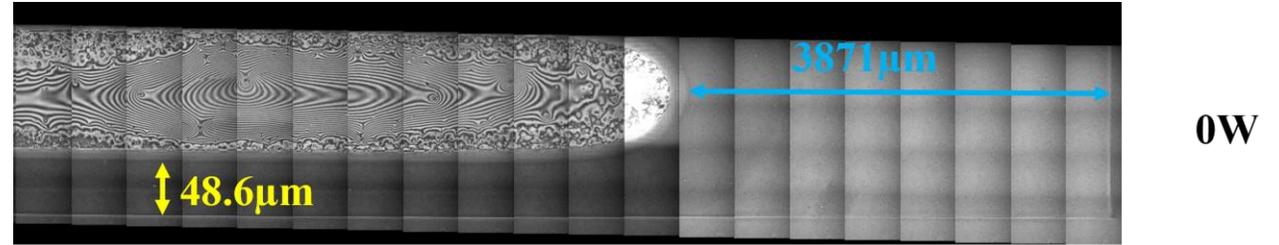
Correlation with experiments

Heater (W)	α_c	α_{TST}	%area increase	α_c^{new}
0.4	0.824	0.770	6.8	0.812
0.6	0.888	0.765	17.1	0.722

Underestimation of interfacial area



Overestimation of α_c



Discontinuity in film profile



Summary/Conclusions

- A multiscale model for phase change during the CVB experiments shows non-uniform interfacial evaporation/condensation.
- This leads to flow recirculations as a result of Marangoni forces.
- These flow recirculations are correlated with heater-side flooding observed at higher heater settings.
- A 3.2% change in β can lead to errors $>50\%$ for the calculation of the evaporation and condensation coefficients.

Acknowledgements

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